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Design and Preliminary Testing of a Magnetic Spring as an Energy-Storing System for Reduced Power Consumption of a Humanoid Arm

Jhon F. Rodríguez-León ¹^(b), Ilse Cervantes ¹, Eduardo Castillo-Castañeda ¹^(b), Giuseppe Carbone ²^(b) and Daniele Cafolla ^{3,*}^(b)

- ¹ Instituto Politécnico Nacional-CICATA Querétaro, Querétaro 76090, Mexico;
- jrodriguezl1600@alumno.ipn.mx (J.F.R.-L.); icervantesc@ipn.mx (I.C.); ecastilloca@ipn.mx (E.C.-C.)
- ² DIMEG, University of Calabria, 87036 Rende, Italy; giuseppe.carbone@unical.it
- ³ Biomechatronics Laboratory, IRCCS Neuromed, 86077 Pozzilli, Italy
- * Correspondence: contact@danielecafolla.eu



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Copyright: © 2021 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). Abstract: The increasing use of robots in the industry, the growing energy prices, and higher environmental awareness have driven research to find new solutions for reducing energy consumption. In additional, in most robotic tasks, energy is used to overcome the forces of gravity, but in a few industrial applications, the force of gravity is used as a source of energy. For this reason, the use of magnetic springs with actuators may reduce the energy consumption of robots performing trajectories due their high-hardness magnetic properties of energy storage. Accordingly, this paper proposes a magnetic spring configuration as an energy-storing system for a two DoF humanoid arm. Thus, an integration of the magnetic spring system in the robot is described. A control strategy is proposed to enable autonomous use. In this paper, the proposed device is modeled and analyzed with simulations as: mechanical energy consumption and kinetic energy rotational and multibody dynamics. Furthermore, a prototype was manufactured and validated experimentally. A preliminary test to check the interaction between the magnetic spring system with the mechanism and the trajectory performance was carried out. Finally, an energy consumption comparison with and without the magnetic spring is also presented.

Keywords: magnetic spring; energy consumption; humanoid arm; control strategy; energy-storing system; neurorehabilitation

1. Introduction

Humanoid robots have been a source of study for at least two decades, with applications in industry, medicine, and, especially, in the area of rehabilitation [1,2]. Many of these devices are based on robots of serial and parallel architectures [3]. However, these robotic devices present several problems for their development, and one of them is a high energy consumption during rehabilitation tasks. One way to achieve an energy consumption reduction is a correct balancing strategy. There have been many balancing methods developed over the years, all of them achieving their principal goal, which is balancing robots. These methods were designed according to the necessities of each robot. In reference [4], the author made a review of the mechanical gravity compensation and balancing solutions. He divided the solutions into three general groups, which are: gravity compensation by counterweights, by auxiliary actuators, and balancing by springs. The main principle of the first technique is adding extra mass (counterweights) to the moving links to keep their center of mass balanced. An example of this type was developed in reference [4]; the author harnessed the mass of the hydraulic robot system to balance itself. Industrial robots KUKA R630 and PUMA 200 use a similar system but harness their electric motor mass. The second type uses auxiliary links (a mechanical system between

the initial structure and the balancing element [5]) for balance. An example of this type was carried out in reference [6], where the authors applied a novel auxiliary linkage system to a five Degrees of Freedom (DoF) Penta-G robot. The next group of balancing solutions has several subtypes, because there are different types of auxiliary actuators. An example is the cable and pulley arrangement, which uses the simple principle of sliding a cable through a pulley to move a mass. In reference [7], the authors applied the current technique to a glove exoskeleton for obtaining and transmitting the data of the hand gestures to a mobile device. A pneumatic or hydraulic system can be used for balancing a robot. In reference [8], the authors developed a mechatronic concept using a gas spring; the gas exerted a quasi-constant force against the external force applied to it. This system achieved a balance into a human-operated load to reduce the human effort while operating it.

The balancing by springs technique has been relevant in recent years, since it brings more benefits than the other strategies [6]. There are two types of springs, the zero free length and the non-zero free. The zero free length is a spring that exerts zero force at zero length, but this is mechanically complex, since a coil will never be a zero length. On the other hand, the non-zero free length exerts zero force at nonzero positions [6]. In reference [9], the authors used a novel gravity compensation system using springs for a four Degrees of Freedom (DoF) manipulator that had parallelogram links. They focused on reducing the gravity torque in the joints by adding springs. Furthermore, in reference [10], the authors proposed a system using nonlinear springs for gravity balancing. They were applied to a robotic arm with an elbow, achieving a reduction of the actuators torques. The work presented in reference [11] defined that an energy consumption reduction can be achieved by modifying the hardware and the software of a robot. The hardware is divided into three categories: robot type, hardware replacement, and hardware addition. This subclass refers to analyzing what kind of robot would be adequate for the required work tasks and for achieve energy savings. In reference [12], a study was carried out to find out which type of robot would be more efficient between parallel and serial structures. According to this study, the parallel one (horizontal position) achieved a better average performance than the other ones. The next hardware category is the hardware replacement; this type looks for a weight reduction in the robot by making its arms or joints as light as possible, so, in that way, the robot can move with less power [11]. An example is the DLR's torque-controlled lightweight robot III, where the authors made an effort by improve their ultralight carbon-fiber arms in their robot, achieving less energy loss [13]. The last hardware type is the hardware addition. This is a relative novel technique. Its principal objective is to use additional hardware as energy-storing and recovery devices. It can be divided into two main groups, which are: Energy-Storing-Device Types and the Energy-Sharing Devices [11]. According to reference [14], the Energy-Storing-Device Types can be divided into four types of kinetic-energy recovery systems (KERS): mechanical KERS, electric KERS, hydraulic KERS, and hydroelectric KERS. Focusing on the mechanical KERS type: it uses mechanical devices for energy storage. In reference [15], these devices were defined as "mechanical energy capacitors. One type of this mechanical capacitors" is the elastic elements, and the famous elastic element is the springs.

In reference [16], the authors designed an energy-saving system using springs for a DELTA robot focusing on pick-and-place operations. They used a multibody simulation to find out the optimal spring specs and the optimal trajectory to harness the robot's natural dynamics. They collocated the springs in parallel with the electric motors; with this, the springs provided the required torque for moving the links, reducing the amount of the exerted torque from the actuators. Another system was developed in reference [17]; the authors made a hybrid system to harness the energy-storing devices and energy-recovering devices to get both benefits. In reference [18], the authors also used springs for energy consumption reduction in the pick-and-place applied to a Reconfigurable-Planar-Linkage (RePlaLink), obtaining an energy consumption reduction in contrast to a similar robot but without the springs. In reference [19], a highly efficient pick-and-place technique was developed by the authors for SCARA robots. They used a similar configuration as in

reference [17] for the springs. The authors set arbitrary initial and end positions for the study, because the robot was not fixed in one position. The technique uses magnets instead of springs, gears, pneumatics, or another kind of auxiliary system. Usually, it is compared with springs since it brings a similar performance but with some advantages over springs, like noncontact, zero vibrations, zero noises, etc.

In this paper, a novel magnetic spring strategy is proposed and added to a two Degrees of Freedom (DoF) serial robot model. A novel configuration and experimental characterization of the novel magnetic spring configuration is introduced. In addition, to identify the initial position of the end effector (EE) at the startup in the robot, the control algorithm is improved. Finally, an energy consumption comparison, with and without a magnetic spring, is described.

2. Conceptual Design and Dynamic Model of a Two DoF Humanoid Arm

The dynamic model of a serial robot with two Degrees of Freedom (DoF) (Double Pendulum) is presented, which simulates the action of the human arm, composed of two joints variables (q_1 and q_2). Figure 1 shows the motion ranges of each joint. The model allows calculating the necessary torque value during the different trajectories, in addition to modifying the distance of each of its links, masses, angles of rotation, speeds, and accelerations (see Figure 2a). The humanoid arm can be used as a prosthetic and neurorehabilitation; in Figure 2b, the CAD model is shown.



Figure 1. Model of the human arm and elbow joints during flexion and extension movements.



Figure 2. Proposed model: (a) mechanism (b) CAD model.

The mechanism is composed of two active joints, q_1 and q_2 , that represent the shoulder and elbow joint, respectively, as well as a series of variables to control. The data for an average human arm [20,21] are presented in Table 1 below.

Table 1. Data from an average human arm [22].

	2 1 2 1	
Mass	$m_1 = 2.13 \text{ kg}$	$m_2 = 1.49 \text{ kg}$
Links	$L_1 = 0.316 \text{ m}$	$L_2 = 0.265 \text{ m}$
Distance between the axes of rotation and the centers of mass	$L_{c_1} = 0.162 \text{ m}$	$L_{c_2} = 0.127 \text{ m}$
Moments of inertia Y-axis	$I_1 = 1.66 \text{ kgm}^2$	$I_2 = 1.66 \text{ kgm}^2$
Friction constant	$K_1 = -1 \text{ N}$	$K_2 = -1 \text{ N}$

The direct and inverse kinematics of the two DoF mechanism were programed to carry out two trajectories: a circle and a lemniscate; these trajectories were tested to evaluate their feasibility and to assist the training exercises for the human arms in references [21,22]. The equations that allow to generate each of the desired trajectories, x_0 and y_{0} , are presented by considering their origin and radius as

$$x_c = 0.2 \text{ m}; y_c = 0.2 \text{ m}; r = 0.1 \text{ m}$$
 (1)

$$x_0 = x_c + rsin(n);$$
 $y_0 = y_c + rcos(n)$ (2)

$$x_0 = x_c + rsin(2n);$$
 $y_0 = y_c + rcos(n)$ (3)

where x_c and y_c are the center of the trajectory, r is the radius, and n varies from 0° to 360°. Equations (2) and (3) are used to perform a circle and the lemniscate, respectively.

The kinematic equations are programmed and simulated in MATLAB R2020b to find the angles of the joints and the positions of the end effector (EE) that allow generating the two previously defined trajectories.

Torque, Energy Consumption, and Electrical Power

The total mechanical energy E_T of the system is a function of the kinetic energy EC_T and the potential energy U_T ; then, the equations that allow obtaining the value of the total energy of the system are

$$E_T = EC_T + U_T \tag{4}$$

$$EC_T = K_1 + K_2 \tag{5}$$

$$U_T = U_1 + U_2 \tag{6}$$

$$K_1 = \frac{1}{2}m_1L_{c_1}^2q_1^2 + \frac{1}{2}I_1q_1^2 \tag{7}$$

$$K_{2} = \frac{m_{2}}{2}I_{1}^{2}q_{1}^{2} + \frac{m_{2}}{2}L_{c_{2}}^{2}\left[q_{1}^{2} + 2q_{1}^{2}q_{1}^{2} + q_{2}^{2}\right] + m_{2}L_{1}L_{c_{2}}\left[q_{1}^{2} + q_{1}^{2}q_{2}^{2}\right]\cos(q_{2}) + \frac{1}{2}I_{2}\left[q_{1}^{2} + q_{2}^{2}\right]^{2}$$
(8)

$$U_1 = -m_1 L_{c_1} \cos(q_1) \tag{9}$$

$$U_2 = -m_2 L_1 g \cos(q_1) - m_2 L_{c_2} g \cos(q_1 + q_2)$$
⁽¹⁰⁾

The total electrical power
$$P_T$$
 is calculated as a function of the consumed torques T_1 and T_2 and the speed of each of the links:

$$P_1 = T_1 q_1^{\cdot} \tag{11}$$

$$P_2 = T_2 q_2^{\cdot} \tag{12}$$

$$P_T = P_1 + P_2 \tag{13}$$

3. A Magnetic Spring as an Energy-Storing System

The magnetic spring system is composed of two identical well-shaped cylindrical magnets, as in Table 2 [23], which shows the magnet proprieties. To calculate the force generated in the magnetic spring, several equations were developed. First, there was the Coulomb's law, which focused on the magnet's charges and the distance between them to obtain the magnetic force [24–27]. Additionally, there was the equation that uses low order polynomials to calculate it [28,29]. Another equation was developed in reference [30], in which the authors defined their formula as the sum of the forces due to the four pairs of the surface.

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Material	NdFeB
Exterior diameter	3/4" (19 mm)
Interior diameter	1/4″ (6.35 mm)
Thickness	1/8" (3.175 mm)
Tolerance	+/-0.1 mm
Mass	4.7 g
Grade	N35
Subjection force	3.3 kg
Magnetization type	Axial

The proposed experimental characterization used two magnet (fixed and float) and a gram scale balance OHAUS model CS2000. U(s) is the distance between the magnets, and Y(s) is the repulsive force calculated using the data of the balance and multiplied to reach a gravity force of 9.81 m/s². The interaction of displacement–force between the magnets in the function to the time is shown in Figure 3.



Figure 3. Interaction of displacement–force between magnets.

The System Identification toolbox from MATLAB was used to estimate the coefficients and the order of a transfer function, a third-order system with zero, as

$$G(s) = \frac{Y(s)}{U(s)} = \frac{-4.727 \times 10^{-1} s + 5.397 \times 210^{-2}}{s^3 + 5.904 \times 10^{-1} s^2 + 0.01121 s + 1.794 \times 10^{-6}}$$
(14)

4. Integration of the Magnetic Spring System in the Robot

According to the previous information, a configuration that integrated the magnetic spring system with the mechanism was proposed, as in Figure 4a,b. In addition, the parameters that define this configuration are presented in Table 3. The objective is to provide energy to the actuation system based on the magnetic field principle.



Figure 4. Displacement–force scheme between magnets in a mechanism: (**a**) diagram, (**b**) conceptual sketch.

where τ_{e_1} and τ_{e_2} are the external torques generated by the magnetic spring system, M_{s_1} and M_{s_2} are the fixed magnets, M_{m_1} and M_{m_2} are the float magnets, d_{m_1} and d_{m_2} are the distances between magnets fixed and float, respectively, F_1 and F_2 are the repulsive forces between the magnets, L_{x_1} and L_{x_2} are the positions of the fixed magnets around L_1 and L_2 , and L_t is the max distance between articulations to the fixed magnet.

Table 3. Design parameters by integration of the magnetic spring system in the robot.

Variable	Min Value	Max Value
d_{m_1}	0.003 m	0.028 m
d_{m_2}	0.003 m	0.028 m
F_1	0.130 N	5 N
F_2	0.130 N	5 N
L_{x_1}	0.010 m	L_1
L_{x_2}	0.010 m	L_2

Figure 4a,b allows describing τ_{e_1} and τ_{e_2} when interacting with the mechanism, and Table 3 shows the parameters of the system. Therefore, the equations to calculate τ_{e_1} and τ_{e_2} generated by the magnetic spring system, can be expressed as

$$d_{m_i} = \sqrt{\left|M_{s_{y_i}} - M_{m_{y_i}}\right|^2 + \left|M_{s_{x_i}} - M_{m_{x_i}}\right|^2} \tag{15}$$

$$F_{i} = \frac{f \ M_{s_{i}} \ M_{m_{i}}}{d_{m_{i}}^{2}} \tag{16}$$

$$q_i = \sin^{-1} \left(\frac{d_{m_i}}{L_{x_i}} \right) \tag{17}$$

$$\tau_{e_i} = F_i L_{x_i} i = \{1, 2\} \tag{18}$$

where $M_{s_{y_i}}$, $M_{m_{y_i}}$, $M_{s_{x_i}}$, and $M_{m_{x_i}}$ are the coordinates x and y of the magnets, and f is the magnetic field constant.

The analysis of the mechanical energy and electrical power consumption of the dynamic model of the robot is based on the implementation of the Proportional Derivative (PD) control with Desired Gravity Compensation (PD-DGC) [31,32].

5. Control Strategy and Simulations

The aim of the pure position control of manipulative robots whose dynamic models include the gravitational torque term $g(q_{d_i})$ is solved by the PD-DGD [31]. The PD-DGD control law is represented as

$$\tau_i = k_{p_i} q_{e_i} + k_{v_i} q_{e_i} + g(q_{d_i}) \quad i = \{1, 2\}$$
(19)

where τ_1 and τ_2 are the joints torques; $k_{p_1} = 120$, $k_{p_2} = 81$, $k_{v_1} = 35$, and $k_{v_2} = 15$ are the proportional and derivative constants, respectively, and are the positive-definite matrix, q_{e_1} and q_{e_2} are the trajectories errors, as in Equation (20), and $g(q_{d_1})$ and $g(q_{d_2})$ are the torques due to the gravity forces in positions q_{d_1} and q_{d_2} [32]. The only difference with the gravity-compensated PD controller is that the term $g(q_{d_i})$ replaces $g(q_i)$. The practical convenience of this controller is evident when the desired position q_d is periodic or constant. Indeed, the vector $g(q_{d_i})$, which depends on q_d and not on q_i , can be evaluated "offline" once q_d has been defined and, therefore, it is no longer necessary to evaluate $g(q_i)$ in real time.

$$q_{e_i} = (q_{d_i} - q_i) \quad i = \{1, 2\}$$
(20)

Figure 5 shows the two trajectories generated by the mechanism by means of a PD-DGD-type control algorithm with the desired gravity compensation; in addition, the start and end of the trajectory located at $q_1 = 0^\circ$ and $q_2 = 0^\circ$ are presented.



Figure 5. Simulation of the trajectories by implementing the PD-DGD-type control algorithm with compensation of the desired gravity: (**a**) circle, (**b**) lemniscate.

The block diagram in Figure 6 describes the elements to implement the control strategy of the two DoF robot arm, including the magnetic springs.

The torque generated by the magnetic springs are added to the torque generated by the robot arm actuators by the control algorithm τ_{c_i} , and the torque generated by the magnetic spring system τ_{e_i} is added and enters the dynamic model of the robot to follow the trajectory (position and velocity). In addition, through this control diagram, it is possible to know when the torque generated by the magnetic spring system acts on the system, which indicates when the system provides energy to the control.



Figure 6. Control diagram, including the magnetic spring system.

5.1. Mechanical Energy Consumption Simulation

To analyze the power consumption of the robot arm with the magnetic spring system during the trajectory, a mechanical energy consumption simulation was carried out. In Figure 7a–c, three states are observed: initial position, trajectory generation, and return to the initial or final position. In addition, the total mechanical energy consumption E_c (J) of the system is shown for each of the mentioned states. The generation of the circular trajectory is programmed by implementing a PD-DGD-type control algorithm with gravity compensation.

Figure 8 shows the required mechanical energy in the robot during the circular trajectory from the starting to the 400-sample time. The red line corresponds to a mechanical energy consumption without the magnetic spring system, and the blue line corresponds to a mechanical energy consumption with the magnetic spring.

The results in Figure 8 show a significant reduction on the mechanical energy that the control must apply to the robot. The estimation of the energy consumption during the circle trajectory is reduced by 71.47% with the magnetic spring system.

5.2. Kinetic Energy Rotational Simulation

To validate the power consumption, a kinetic energy rotational simulation is proposed; the simulation was developed in Working Model (version 5.1.2.53, San Mateo, CA, USA). The objective was to measure the kinetic energy rotational from $q_2 = 0^\circ$ to $q_2 = 15$ with and without the magnetic force. In the scenario with the magnetic force applied the actuator, torque was input as 20 Nm, and a 5-N force along the x-axis was applied for 0.5 s, simulating the magnetic field. In the scenario without the magnetic force, only the actuator torque was input as 25 Nm. Both simulation scenarios lasted 1 s. The frictionless motion of L_2 was programmed, and the gravitational force was included. The simulation parameters of the mechanism are shown in Table 1. Figure 9 a and b shows a scheme of both the simulated scenarios.



Figure 7. Simulation of the 3 states that allow generating the circular trajectory by implementing the PD-DGD-type control algorithm with gravity compensation: (**a**) startup, (**b**) trajectory, (**c**) return to startup.







Figure 9. Power consumption simulation scheme: (a) with magnetic force, (b) without magnetic force.

Figure 10a,b shows the power consumption with and without the magnetic force; the measure of the kinetic energy rotational with a magnetic force was 0.5 J, and the measure of the kinetic energy rotational without a magnetic force was 0.9 J. The results in Figure 10a,b show a significant reduction as 0.4 J on the kinetic energy rotational applied to the mechanism. Therefore, when the magnets of the system are employed, the motor only needs to overcome the dissipative forces, reducing the energy consumption.



Figure 10. Power consumption: (a) with magnetic force, (b) without magnetic force.

5.3. Multibody Dynamic Simulation

To analyze the interaction between the mechanisms with the magnetic spring system for the flexion and extension trajectories, a multibody dynamic analysis was carried out. For the simulation, a trajectory from $q_2 = 0^\circ$ to $q_2 = 15^\circ$ ° was programmed for 5 s to move the mass of $L_2 = 1121.48$ g and material PLA. An external force generated by the magnetic spring system of 5 N was applied after 3 s from the start of the trajectory; this force was applied on top of the surface of the magnet in a direction normal to the surface of the magnet.

The torque applied by the mechanism before and after interacting with the external force of the magnetic spring system was analyzed, as shown in Figure 11a. The results show that, at the beginning of the trajectory, a torque of 242.15 Nm was needed, which increased until it reached 349.65 Nm in 3 s; after the external force was applied, the torque decreased to 125.57 Nm until it reached 48.06 Nm in 5 s. Therefore, at 3 s, the torque decreased 224.08 Nm due to the interaction of the magnetic spring system.



Figure 11. Multibody simulations: (a) torque, (b) power consumption.

In Figure 11b, the analysis of the energy consumption is presented, and the data shows that, at the beginning of the trajectory, 0.0120 W was consumed, which increased until reaching 0.0190 W in 3 s; then, when the magnetic force was applied, the energy consumption went down to -0.0065 W in 3 s. This result indicates that the magnetic

system causes the mechanism to behave like a generator system, since it provides more energy than it consumes. Therefore, this result, due to the magnitude of the external force generated by the magnetic system, is higher than that required by the mechanism.

6. Prototype Manufacturing and Experimental Test

After the design and simulation of a magnetic spring as an energy-storing system for a two DoF humanoid arm, a prototype was produced with additive manufacturing [33]. The geometrical parameters of the mechanism with the magnetic spring system are shown in Table 4. Two magnet N45 used for the magnetic-spring system, two Dynamixel AX-12A servomotors motors, were chosen for the actuation system of the prototype, as shown in Figure 12. The controller is composed of an Arduino Mega board that is connected to two motors, and this board allows speed and direction control of the motor at the same time. The power supply generates 5 V DC to 12 V DC and a maximum current of 2 A. The prototype weight is 2.5 kg, and thanks to the additive manufacturing technology used, the manufacturing price of the prototype is less than \$400.

Table 4. Geometrical parameters of a magnetic spring as an energy-storing system for a two DoF humanoid arm.

Dimension	Value (mm)
L_1	250
L_2	170
L_{x_1}	60
L_{x_2}	120



Figure 12. Prototype with the magnetic spring system: (a) forearm detail, (b) full arm detail, (c) arm detail.

Preliminary tests were carried out to check the interaction between the magnetic spring system with the mechanism and the trajectory performance. The aim of the experiment was to measure the ability of the mechanism to generate the flexion and extension trajectories in the end effector (EE) defined by the x₁ and y₁ axes. Thus, two poses were programed to test the reaction of the system. These poses consisted of a flexion and extension trajectory defined from $q_2 = 0^\circ$ to $q_2 = 90^\circ$ and $q_2 = 90^\circ$ to $q_2 = 0^\circ$, as shown in Figure 13a–c, respectively.



Figure 13. Flexion and extension trajectories: (a) capture at 0° , (b) capture at 45° , (c) capture at 90° .

To validate the trajectory performance, a strategy on marker detection was used to identify the EE position through image processing; this strategy was explained in reference [34]. Figure 14a–c shows the EE identification in green color, and Figure 15 shows the comparison between the program trajectory and the trajectory generated by the EE, defined by x_i and y_i .



Figure 14. Identification of the trajectory: (a) capture at 90° , (b) capture at 45° , (c) capture at 0° .



Figure 15. Compares between the program trajectory and the trajectory generated by the EE.

The results in Figure 15 show a significant error in the trajectory that the mechanism generated. The estimation of the trajectory error is 3%.

To validate the low power consumption of the mechanism with and without the magnetic spring system, an experiment was proposed to measure the current consumption. The current consumption was measured for a trajectory from $q_2 = 0^\circ$ to $q_2 = 15^\circ$. This trajectory was selected to function with the specifications of the magnetic system shown in Table 3. A T-RMS digital multimeter powered by Steren was used. The results obtained showed that the mechanism with the magnetic spring system consumed a current at the startup of the trajectory of 0.031 A and finished at 0.113 A. On the other hand, when the mechanism was without the magnetic spring system, the current consumption at the startup of the trajectory was 0.050 A and finished 0.140 A. Therefore, it can be validated that the current consumption in the startup was 0.019 A less by using the magnetic spring system. The results obtained are presented in Figure 16a,b. The average current consumption with and without the magnetic force was 11.46 A and 13.81 A, respectively. Considering that the voltage was the same, this results in about a 17% power consumption reduction when introducing the magnetic force; this magnetic force has a production cost of approximately \$12.



Figure 16. Current consumption: (a) with magnetic force, (b) without magnetic force.

All in all, the focus of this paper was on the design of a magnetic spring as an energy-storing system, integration of the magnetic spring with the humanoid arm, and preliminary testing for reduced power consumption. The advantages with other designs, as shown in reference [16], are that they use physical contact, and we proposed no contact; this allowed us to avoid mechanical wear between the moving parts of the prototype. However, we had limitations, such as adding weight to the mechanism, as represented by the magnetic system.

The mechanical design of q_2 that used the additional supports necessary for the magnetic system allowed us to generate the flexion and extension motions of the elbow without limitations. However, for q_1 it was not possible to generate the motions of the shoulder, since the mechanical design had one DoF.

As for future works, the proposed design can be adapted to generate reference trajectories for three-dimensional exercises. In addition, there may be alternative ways to better use a magnetic spring as an energy-storing system for reduced power consumption.

7. Conclusions

In this paper, a magnetic spring as an energy-storing system for a two DoF humanoid arm was proposed. The proposed design was based on the characteristics of the magnetic spring obtained through a displacement–force relationship. The new prototype is characterized by a kinematic model, trajectory performance analysis, and simulations that include a mechanical energy consumption and kinetic energy rotational and multibody dynamics. Then, a low-cost prototype presented its manufacturing process through 3D printing and commercial components. The hardware and software for the control system were then detailed, and experimental tests validated the performance and low power consumption of the proposed mechanism design.

In conclusion, a magnetic spring as an energy-storing system for a two DoF humanoid arm allowed to obtain low energy consumption through the phenomenon of attraction and repulsion of the magnetic field and the action force that compresses them. In future developments, the components of the magnetic spring system will be manufactured with an improved repulsion and attraction force through the electromagnetic models, improving the magnetic force and creating a high energy-efficiency potential.

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Data Availability Statement: The data presented in this study are available on request at the discretion of the corresponding author.

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