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# Research on an Adaptive Variational Mode Decomposition with Double Thresholds for Feature Extraction

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**Abstract:** A motor bearing system is a nonlinear dynamics system with nonlinear support stiffness. It is an asymmetry system, which plays an extremely important role in rotating machinery. In this paper, a center frequency method of double thresholds is proposed to improve the variational mode decomposition (VMD) method, then an adaptive VMD (called DTCFVMD) method is obtained to extract the fault feature. In the DTCFVMD method, a center frequency method of double thresholds is a symmetry method, which is used to determine the decomposed mode number of VMD according to the power spectrum of the signal. The proposed DTCFVMD method is used to decompose the nonlinear and non-stationary vibration signals of motor bearing in order to obtain a series of intrinsic mode functions (IMFs) under different scales. Then, the Hilbert transform is used to analyze the envelope of each mode component and calculate the power spectrum of each mode component. Finally, the power spectrum is used to extract the fault feature frequency for determining the fault type of the motor bearing. To test and verify the effectiveness of the DTCFVMD method, the actual fault vibration signal of the motor bearing is selected in here. The experimental results show that the center frequency method of double thresholds can effectively determine the mode number of the VMD method, and the proposed DTCFVMD method can accurately extract the clear time frequency characteristics of each mode component, and obtain the fault characteristics of characteristics; frequency, rotating frequency, and frequency doubling and so on.

**Keywords:** variational mode decomposition; signal analysis; time-frequency analysis; center frequency method of double thresholds; Hilbert transform

## 1. Introduction

Time-frequency analysis method has been the most commonly applied method in the field of fault diagnosis of rotating machinery [1–3]. Vibration signals of rotating machinery usually contain a lot of information about the equipment health states. During equipment operation, when the

equipment health state takes a turn for the worse, the impact and impact pulses of the equipment will increase [4–6]. Therefore, it is very significant to deeply analyze vibration signals for health monitoring and fault prediction of rotating machinery.

Due to the complex structure and changeable working conditions of rotating machinery, the vibration signals often take on the characteristics of multi-component and non-stationary. The time-frequency analysis method is widely applied to process the vibration signals of rotating machinery in order to extract fault features by using signal decomposition and filtering [7–16]. There exists a lot of time-frequency analysis methods, such as the Gabor transform, wavelet transform, Hilbert-Huang transform, empirical mode decomposition (EMD), local mean decomposition (LMD), empirical wavelet transform (EWT), and so on [17–32]. These time-frequency analysis methods can extract fault features from vibration signals, but they have their own problems in vibration signal analysis. The wavelet transform needs to select the wavelet basis function in advance, thus it is a non-adaptive signal analysis method. The applications of EMD and LMD are restricted because of the existence of mode mixing. Although the EEMD and ensemble LMD methods alleviate the mode mixing problem to a certain extent, the computation complexity is sharply increased and the added white noises cannot be removed. Variational mode decomposition (VMD) is an adaptive and non-recursive signal decomposition method developed by Dragomiretskiy and Zosso in 2014 [33]. The VMD method is used to transform the mode decomposition problem into a variational solution problem. Additionally, the alternating direction multiplier method is used to optimize the VMD. The signal is decomposed into a  $k$  discrete number of sub-signals; near the corresponding center frequency, each component is considered to be tight. The constrained variational optimization problem is used to evaluate the bandwidth of the component. In the optimization, it can obtain a set of mode ensemble with the limited band feature. It is essentially a set of adaptive Wiener filter bank and can separate the modes with different center frequencies. Therefore, the VMD method is widely used to extract the principal modes for fault diagnosis in bearings' signals. Wang et al. [34] studied the equivalent filtering characteristics of VMD, and proposed a rub-impact fault detection method for realizing rotor-stator fault diagnosis. Yi et al. [35] proposed a novel method of fault feature extraction based on the combination of VMD and the particle swarm optimization (PSO) algorithm. Liu et al. [36] proposed a novel signal denoising method that combines variational mode decomposition and detrended fluctuation analysis. Zhang et al. [37] proposed a novel hybrid fault diagnosis approach by using variational mode decomposition and majorization-minimization based total variation denoising for the denoising and non-stationary feature extraction. Liu et al. [38] proposed an improved variational mode decomposition for time-frequency analysis of fault diagnosis of a turbine rotor. Henaoui et al. [39] reviewed diagnostic techniques for electrical machines. An et al. [40] proposed a gear fault diagnosis method based on variational mode decomposition and envelope analysis according to the modulation characteristics of the gear vibration signal arising from faults therein. Mahgoun et al. [41] proposed a gear fault detection method by using the VMD method at a variable rotating speed. Abdoos et al. [42] proposed a new hybrid algorithm based on VMD, S-transform, and support vector machines for power quality disturbances' detection in electrical power systems. Li et al. [43] proposed an independence-oriented VMD method for identifying the fault feature of the wheel set bearing of a high-speed locomotive. From the application of the VMD method, it can be seen that the VMD method can better extract the multiple features from a vibration signal. However, the VMD method needs to give the mode number in advance. If the given mode number is unreasonable, the VMD method will cause the loss of important modes. Now, the VMD method is widely used to analyze the signal, and the central frequency observation method is used to determine the appropriate mode number. In this method, a number of values of the mode number is preset to repeatedly run VMD several times, and then the results of all trials are observed to determine the appropriate value of the mode number. This method requires manual judgments and experience to determine the appropriate value of the mode number, thus it will greatly limit the adaptability of VMD method. In order to overcome this drawback, some researchers have deeply studied and improved

the VMD method. Immovilli and Cocconcelli [44] investigated shaft radial load effect on bearing fault signatures' detection. Wang et al. [45] proposed a complex variational mode decomposition method for the analysis of complex-valued data. Zhang et al. [46] proposed an improved VMD method with adaptive parameter selection to analyze vibration signals. Mohanty et al. [47] proposed a novel fault identification method based on combining variational mode decomposition, the Hurst exponent, and correlation coefficient. Choi et al. [48] proposed an improvement of variational mode decomposition that can effectively handle problems caused by missing values. Wang et al. [49] proposed a quasi-bivariate VMD method for extracting features with various scales. In addition, the other feature extraction methods are proposed in recent years [50–52].

These improved VMD methods can better determine the mode number and extract the multiple features from a vibration signal. However, some limitations remain regarding the adaptability of the VMD method. Therefore, it is necessary to continue to deeply study the appropriate mode number in the VMD. In this paper, a method called VMD with the center frequency method of double thresholds (DTCFVMD) is proposed to determine the appropriate mode number. Then, the DTCFVMD method is introduced into signal analysis to identify the bearing fault type. Finally, the actual fault vibration signal of the motor bearing is selected to test and verify the effectiveness of the DTCFVMD method.

## 2. Basic Methods

### 2.1. Variational Mode Decomposition (VMD) Method

The VMD method is an adaptive signal decomposition method by Dragomiretskiy and Zosso [33]. It is used to decompose a real signal into a set of sub-signals. Near the corresponding center frequency, each component is considered to be tight. The constrained variational optimization problem is used to evaluate the bandwidth of the component. Finally, in the optimization, the VMD method can obtain a solution for the constrained variational problem.

In the VMD method, the intrinsic mode function (IMF) is defined as an amplitude modulation-amplitude frequency (AM-FM) signal. Its expression is described as follows:

$$u_k(t) = A_k(t) \cos(\varphi_k(t)) \quad (1)$$

where,  $\varphi_k(t)$  is a non-decreasing function,  $\varphi_k'(t) \geq 0$ ,  $A_k(t)$  is the envelopment; the change of the envelopment,  $A_k(t)$ , and the instantaneous frequency,  $w_k(t) = \varphi_k'(t)$ , is much slower than the phase,  $\varphi_k(t)$ . In other words, in sufficiently long intervals,  $[t - \delta, t + \delta]$ ,  $\delta \approx 2\pi / \varphi_k'(t)$ ,  $u_k(t)$  can be regarded as a pure harmonic signal with the amplitude,  $A_k(t)$ , and frequency,  $w_k(t)$ .

To evaluate the bandwidth of the mode, for each mode function,  $u_k(t)$ , the Hilbert transform is used to obtain the unilateral spectrum of the signal,  $(\delta(t) + j/\pi t) * u_k(t)$ .  $\delta(t)$  is the distribution function of Dirichlet,  $*$  represents the convolution. An exponential term,  $e^{jw_k t}$ , is added to adjust the estimated center frequency of each mode function of the corresponding analytic signal, and the spectrum of each mode is transferred to the base band,  $[(\delta(t) + j/\pi t) * u_k(t)]e^{jw_k t}$ .  $w_k$  represents the center frequency. Then, the Gauss smoothing of the signal is demodulated to obtain the constraint variational problem.

$$\min_{\{u_k\}, \{w_k\}} \left\{ \sum_k \left\| \partial_t \left[ (\delta(t) + \frac{j}{\pi t}) * u_k(t) \right] e^{-jw_k t} \right\|_2^2 \right\} \quad (2)$$

subject to:

$$\sum_k u_k = f \quad (3)$$

To transform the constraint problem into an unconstrained problem, Lagrangian multipliers,  $\lambda$ , and quadratic penalty term,  $\alpha$ , are applied to render the unconstrained variational problem. The Lagrangian multipliers,  $\lambda$ , can strictly enforce constraints. The quadratic penalty,  $\alpha$ , can ensure the reconstruction accuracy. The augmented Lagrangian is described as:

$$L(\{u_k\}, \{w_k\}, \lambda) = \alpha \left\| \sum_k \partial_t [(\delta(t) + \frac{j}{\pi t}) * u_k(t)] e^{-jw_k t} \right\|_2^2 + \|f(t) - \sum_k u_k(t)\|_2^2 + \langle \lambda(t), f(t) - \sum_k u_k(t) \rangle \quad (4)$$

In VMD, the alternating direction method of the multiplicative operator is used to solve the above variational problem. The mode function,  $u_k^{n+1}$ , center frequency,  $w_k^{n+1}$ , and  $\lambda$  are alternated and updated to seek to expand the saddle point of the Lagrange expression. The value problem of the mode function,  $u_k^{n+1}$ , is described as:

$$u_k^{n+1} = \underset{u_k \in X}{\operatorname{argmin}} \left\{ \alpha \left\| \sum_k \partial_t [(\delta(t) + \frac{j}{\pi t}) * u_k(t)] e^{-jw_k t} \right\|_2^2 + \|f(t) - \sum_k u_k(t) + \frac{\lambda(t)}{2}\|_2^2 \right\} \quad (5)$$

The equidistance transformation of Parseval/Plancherel Fourier is used to transform the formula (4) into the frequency domain.

$$\hat{u}_k^{n+1} = \underset{\hat{u}_k, u_k \in X}{\operatorname{argmin}} \left\{ \alpha \|j(w) [(1 + \operatorname{sgn}(w + w_k)) \hat{u}_k(w + w_k)]\|_2^2 + \|\hat{f}(w) - \sum_i \hat{u}_i(w) + \frac{\hat{\lambda}(w)}{2}\|_2^2 \right\} \quad (6)$$

The  $w$  in the first item is replaced by  $w - w_k$  to obtain the following expression:

$$\hat{u}_k^{n+1} = \underset{\hat{u}_k, u_k \in X}{\operatorname{argmin}} \left\{ \alpha \|j(w - w_k) [(1 + \operatorname{sgn}(w)) \hat{u}_k(w)]\|_2^2 + \|\hat{f}(w) - \sum_i \hat{u}_i(w) + \frac{\hat{\lambda}(w)}{2}\|_2^2 \right\} \quad (7)$$

In the approximation terms of reconstruction, the conjugate symmetry of real signals is used to transform the expression (7) into the form of a non-negative frequency integral.

$$\hat{u}_k^{n+1} = \underset{\hat{u}_k, u_k \in X}{\operatorname{argmin}} \left\{ \int_0^\infty 4\alpha (w - w_k)^2 |\hat{u}_k(w)|^2 + 2|\hat{f}(w) - \sum_i \hat{u}_i(w) + \frac{\hat{\lambda}(w)}{2}|^2 dw \right\} \quad (8)$$

By conversion, the solution of quadratic optimization problem can be obtained as follows:

$$\hat{u}_k^{n+1}(w) = \left( \hat{f}(w) - \sum_{i=k} \hat{u}_i(w) + \frac{\hat{\lambda}(w)}{2} \right) \frac{1}{1 + 2\alpha (w - w_k)^2} \quad (9)$$

where,  $\hat{u}_k^{n+1}(w)$  is equivalent to the wiener filtering of the current residual wiener filtering equivalent to the current residual,  $\hat{f}(w) - \sum_{i=k} \hat{u}_i(w)$ . The full spectrum of the real mode can be completed symmetrically by Hermite symmetry, and the  $\hat{u}_k(w)$  is the real part,  $u_k(t)$ , of the inverse Fourier transform.

The center frequency,  $w_k$ , is the first item of the right side of the expression (5) and the expression (6). Therefore, the updated expression,  $w_k$ , is described as follows:

$$w_k^{n+1} = \underset{w_k}{\operatorname{argmin}} \left\{ \left\| \sum_k \partial_t [(\delta(t) + \frac{j}{\pi t}) * u_k(t)] e^{-jw_k t} \right\|_2^2 \right\} \quad (10)$$

It is like as the mode function,  $u_k$ , and the center frequency,  $w_k$ , in the Fourier domain is optimized as follows:

$$w_k^{n+1} = \underset{w_k}{\operatorname{argmin}} \left\{ \int_0^\infty (w - w_k)^2 |\hat{u}_k(w)|^2 dw \right\} \quad (11)$$

The solution of the above quadratic optimization problem is given:

$$w_k^{n+1} = \frac{\int_0^\infty w |\hat{u}_k(w)|_2 dw}{\int_0^\infty |\hat{u}_k(w)|_2 dw} \quad (12)$$

The algorithm process of VMD is described as the follows:

**Step 1.** Initialize the parameters of VMD, including  $\{\hat{u}_k^1\}$ ,  $\{w_k^1\}$ , and  $n$ .

**Step 2.** Update the  $u_k$  and  $w_k$  according to the expression (9) and expression (12).

**Step 3.** Update the  $\lambda$ .

$$\hat{\lambda}^{n+1}(w) \leftarrow \hat{\lambda}^n(w) + \tau(\hat{f}(w) - \sum_k \hat{u}_k^{n+1}(w)) \quad (13)$$

**Step 4.** Set the error to  $\varepsilon > 0$ . If  $\frac{\sum_k \|\hat{u}_k^{n+1} - \hat{u}_k^n\|_2^2}{\|\hat{u}_k^n\|_2^2} < \varepsilon$  is met, the iteration is terminated. Otherwise, return to Step 2.

## 2.2. Empirical Mode Decomposition and Ensemble Empirical Mode Decomposition

The EMD method is an adaptive decomposition method. The EMD method can decompose non-linear data into a series of amplitude modulated-frequency modulated (AM-FM) components. The EMD decomposes the original signal ( $S(x)$ ) into a number of IMFs:

$$S(t) = \sum_{i=1}^n c_i(t) + r_n(t) \quad (14)$$

where  $r_n(t)$  represents residual error function, and IMF components,  $c_1, c_2, c_3, \dots, c_n$ , contain different elements, respectively, from a low to high frequency of signals.

The EEMD method is an improved EMD method [53,54]. It can solve the mode mixing problem in the VMD method by adding Gaussian white noise. The decomposition principle of the EEMD method is that the additional white noise is uniformly distributed in the whole time-frequency space, and the time-frequency space is composed of different scale components separated by the filter group. When the signal is added with a uniformly distributed white noise, the signal regions under different scales are automatically mapped to the appropriate scales.

## 3. An Adaptive VMD with the Center Frequency Method of Double Thresholds

Compared with the EMD and EEMD methods, VMD is a new signal decomposition method. This method has a better theoretical basis. Its essence is multiple adaptive Wiener filter banks, which take on better noise robustness. In the field of mode decomposition, the VMD can successfully separate two pure harmonic signals with similar frequencies and effectively suppress the mode mixing. However, the VMD decomposition needs to set the number of the decomposition in advance. How to effectively determine the number of the decomposition will directly affect the final decomposition results. If the presupposed value of it is greater than the effective mode number in the signal, it will generate the excessive decomposition phenomenon, decomposing the useless false components to interfere with the analysis of the effective components in the original signal. If the presupposed value is less than the effective mode number in the signal, it will generate the undecomposed phenomenon, which cannot completely decompose the effective component. Therefore, it is a very important role to determine the mode number of the VMD method. The research of the VMD method has just started, and it is a research hotspot topic to determine the mode number for the VMD method at present. There are some methods, which are used to determine the mode number, such as the central frequency method, kurtosis method, and optimization algorithms, and so on. The central frequency value of the corresponding IMF component can be obtained by using the VMD decomposition. The central frequency method can directly determine the mode number by using the central frequency value

obtained by VMD decomposition, so the central frequency method has been widely used. However, the central frequency method depends on the subjective factors when the mode number is selected. Therefore, on the basis of the central frequency method, the dual threshold of the central frequency method is proposed for the adaptive selection mode number in the VMD decomposition. The central frequency method of double thresholds is proposed to adaptively select the mode number in the VMD decomposition.

### 3.1. The Center Frequency Method

#### 3.1.1. The Basic Principle and Implementation Steps

Each IMF component obtained by VMD decomposition corresponds to a central frequency,  $cf$ , and the center frequency of the decomposed IMF component under each scale is distributed from low frequency to high frequency. Set the maximum value,  $k_{\max}$ , of the decomposition scale,  $k$ , and the signal is decomposed under the decomposition scale,  $k = 1, 2, 3, \dots, k_{\max}$ . If the difference between the maximum value of the central frequency of the IMF component under the scale,  $k$ , and the maximum value of the central frequency of the IMF component under the scale,  $k + 1$ , is very small, the IMF component under the scale,  $k$ , is determined to firstly obtain the maximum value of the central frequency. According to the central frequency method, the number,  $k$ , of the VMD decomposition is determined.

The implementation steps of the central frequency method are described as follows:

**Step 1.** Determine the maximum value,  $k_{\max}$ , of the decomposition scale.

**Step 2.** The VMD method is used to decompose the original signal into a series of IMF components under different scales ( $IMF_k (k = 1, 2, 3, \dots, k_{\max})$ ). The central frequency values of each IMF component are calculated to obtain the set,  $\Delta cf_{IMF_k} (k = 1, 2, 3, \dots, k_{\max})$ .

**Step 3.** Calculate the maximum value of the center frequency of the set,  $cf_{IMF_k}$ , under different scales, as  $\Delta cf_{\max_{IMF_k}} (k = 1, 2, 3, \dots, k_{\max})$ .

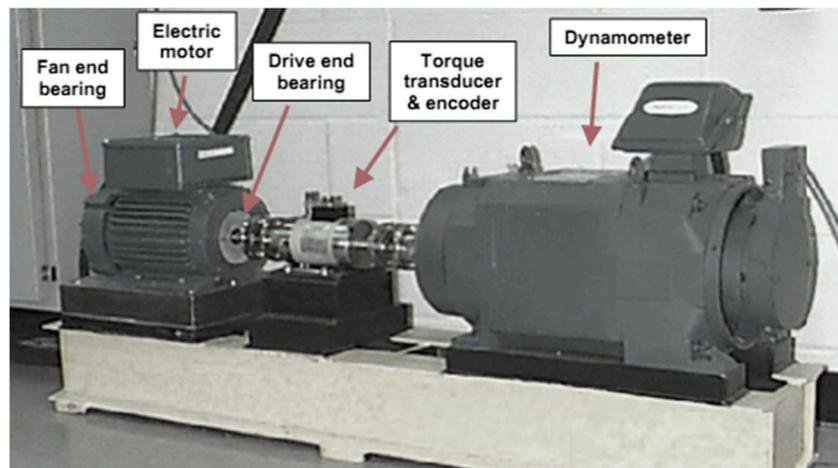
**Step 4.** Calculate the maximum values of the center frequencies of the set,  $\Delta cf_{\max_{IMF_k}}$ , as  $\Delta cf_{\max_{IMF}}$ .

**Step 5.** Calculate the difference between the maximum center frequency value of all modes and the maximum center frequency value of the mode under scale,  $k$ , as  $\Delta cf_{\max_{IMF_k}} = \Delta cf_{\max_{IMF}} - \Delta cf_{\max_{IMF_k}}$ . If there is  $\Delta cf_{\max_k} \ll \Delta cf_{\max_{IMF}}$ , the  $k$  is the mode decomposition number of the VMD method.

When the central frequency method is used to determine the mode number,  $k$ , it is necessary to determine the value,  $k$ , according to the  $\Delta cf_{\max_k} \ll \Delta cf_{\max_{IMF}}$ . In the process, the subjective factors are influenced by human factors. The result will be that the optimal value cannot be obtained to some extent.

#### 3.1.2. Experimental Analysis

The experiment data comes from the Bearing Data Center of Case Western Reserve University [55]. The deep groove ball bearing of 6205-2RS 6 JEM SKF (SKF company, Goteborg, Sweden) was used in this experiment. The experimental platform is shown in Figure 1. The motor with 1.5 Kw was connected to a dynamometer. The accelerometers at the 12 o'clock position on the drive end and fan end of motor housing were used to collect vibration data. The electro-discharge machining method was used to make the bearing's inner race fault and outer race fault. The bearing information bearing is described in Table 1. The vibration signals were obtained under no-load (0 hp) at a rotating speed of 1797 r/min. The fault diameter was 0.1778 mm. The inner race fault and outer race fault vibration signals are selected in here. The vibration signals were sampled at 12,800 Hz. Each vibration signal duration was 10 s. The vibration signals were divided into sample segmentations. Each sample covered 2048 data points.



**Figure 1.** The experimental platform.

**Table 1.** The bearing information of the 6205-2RS JEM SKF deep groove ball bearing.

Inside Diameter	Outside Diameter	Thickness	Ball Diameter	Pitch Diameter	Roller Number	Rotating Speed
25 mm	52 mm	15 mm	8.182 mm	44.2 mm	9	1797 r/min

The theoretical calculation values of the fault characteristic frequency were calculated according to the calculation method, as shown in Table 2.

**Table 2.** Fault characteristic frequencies.

Inner Race	Outer Race	Rolling Element	Switching Frequency
162.2 (Hz)	107.3 (Hz)	141.1 (Hz)	29.2 (Hz)

The inner race and outer race faults of the roller bearing are similar in Figures 2 and 3.

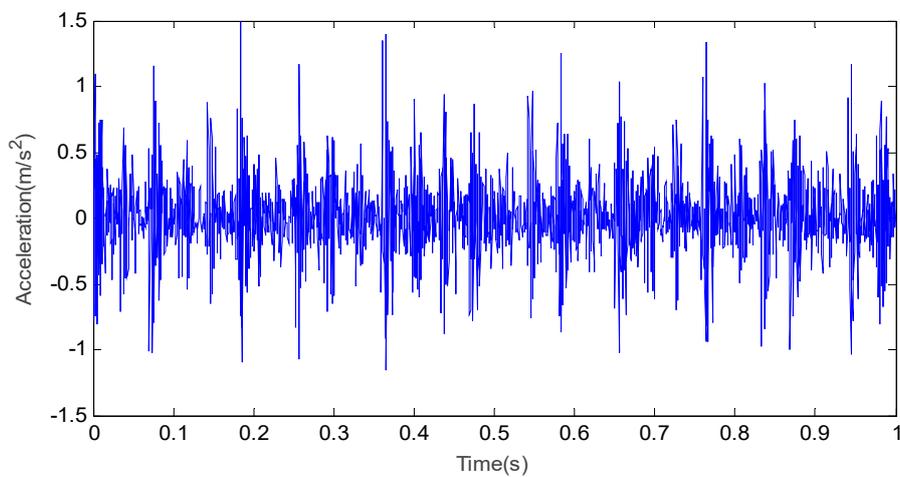
The time domain waveform and frequency domain waveform of the inner race fault vibration signal are shown in Figures 4 and 5.



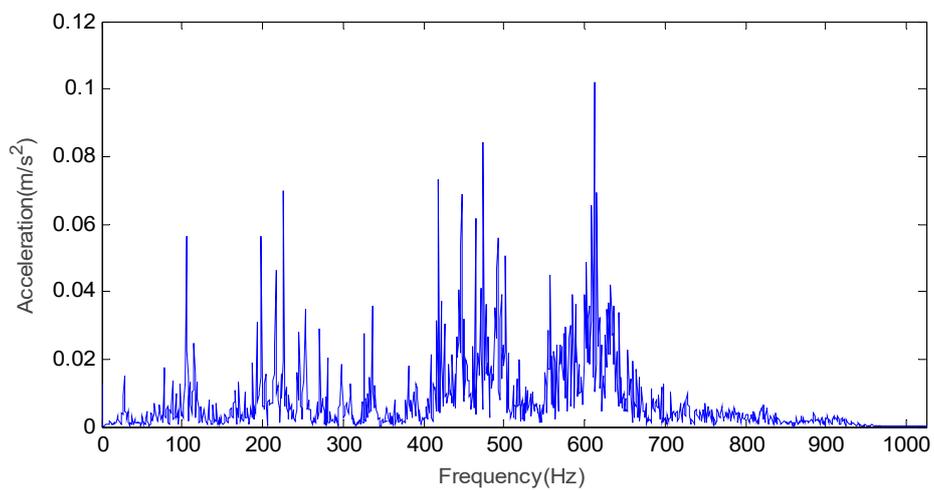
**Figure 2.** The inner race fault of the roller bearing.



**Figure 3.** The outer race fault of the roller bearing.



**Figure 4.** The time domain waveform of the inner race fault vibration signal.



**Figure 5.** The frequency domain waveform of the inner race fault vibration signal.

Firstly, the VMD method was used to decompose the inner race fault vibration signal to obtain the central frequency of each IMF component under different scales, as shown in Figure 6.

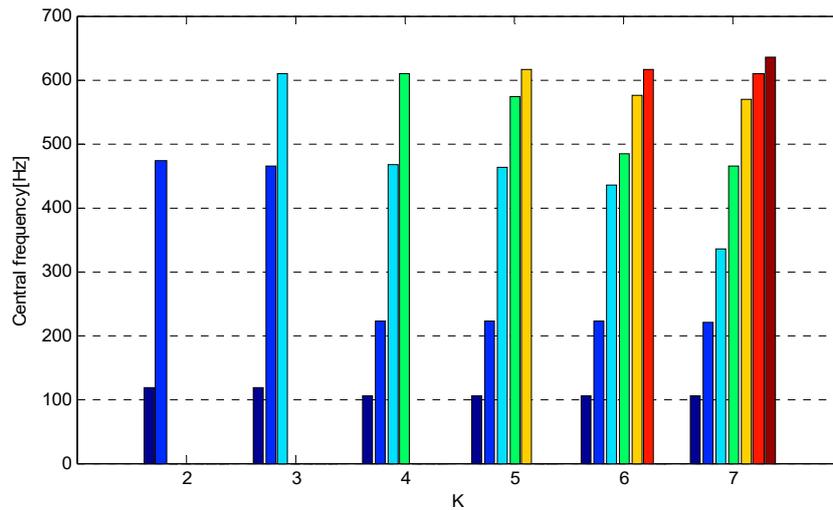


Figure 6. The central frequency of each IMF component under different scales.

As can be seen from Figure 6, when there is  $k = 7$ , the center frequency of the seventh IMF component obtains the maximum value of all IMF components. When there is  $k \geq 3$ , the difference between the maximum value of the central frequency of the IMF component and the maximum value of all IMF components is very small under different scales. According to the center frequency method, the value of  $k$  is determined as  $k = 3$ . When there is  $k \geq 4$ , the new frequency value appears in the mode. The analysis considers that the corresponding mode of the new frequency value is the effective mode. Therefore, when there is  $k = 3$ , the effective mode cannot be completely decomposed. From the experimental results, it can be found that there is an undecomposed phenomenon for determining the value of  $k$  by using the central frequency method.

When the value of  $k$  is 5, the time domain waveform and frequency domain waveform of the inner race fault vibration signal are shown in Figures 7 and 8.

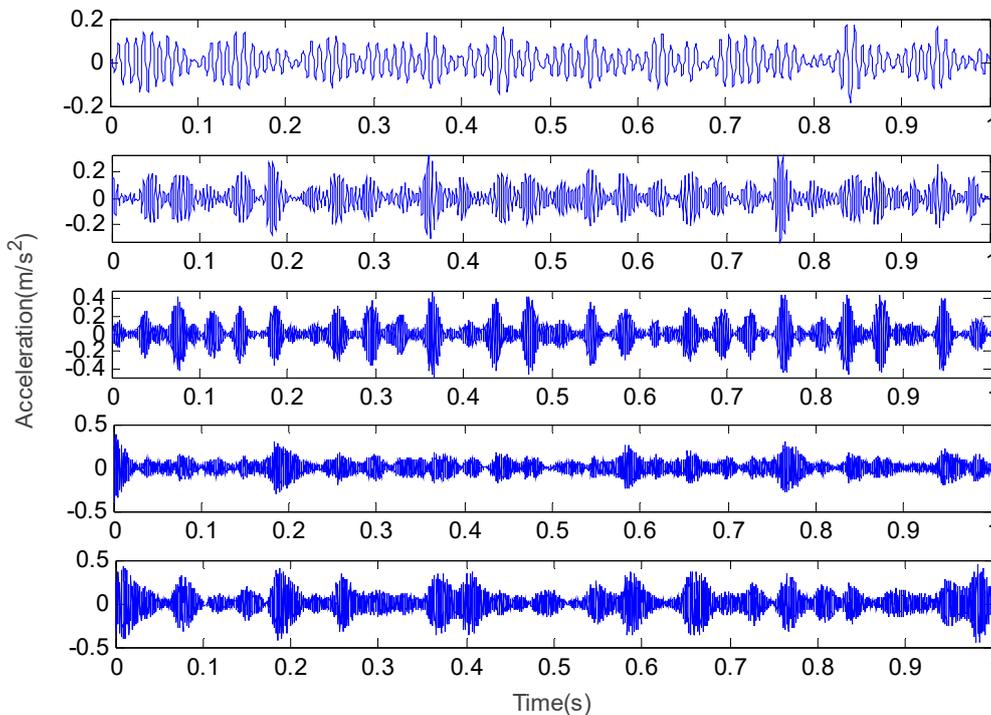
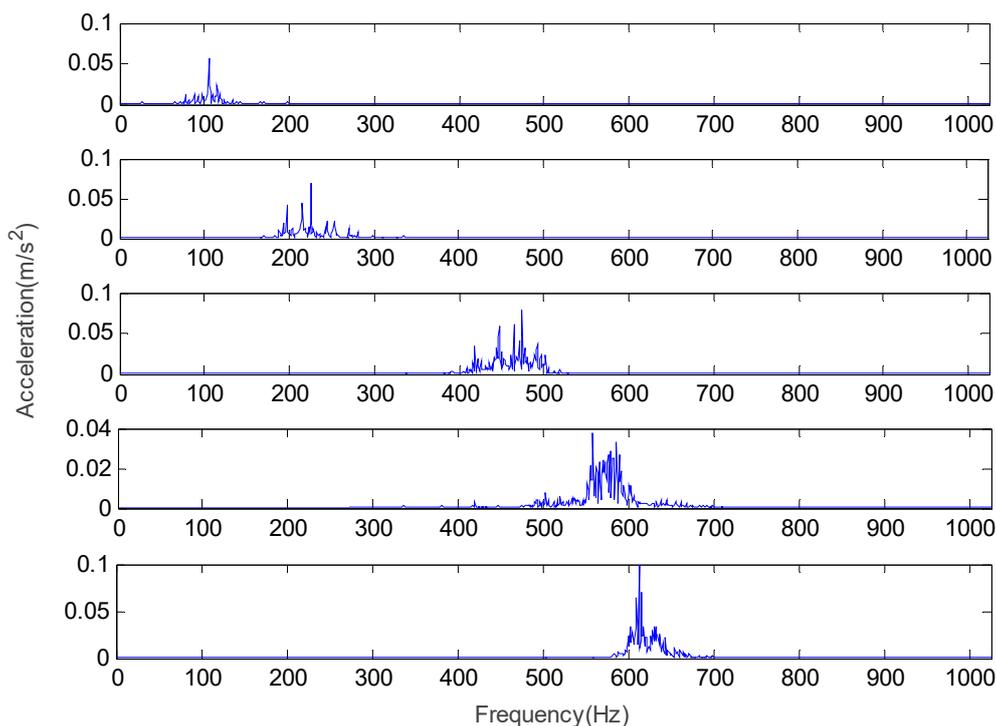


Figure 7. The time domain waveform of each IMF for the inner race at  $k = 5$ .



**Figure 8.** The frequency domain waveform of each IMF for the inner race at  $k = 5$ .

### 3.2. The Center Frequency Method of Double Thresholds

#### 3.2.1. The Idea of the Center Frequency Method of Double Thresholds

For the shortcomings of the center frequency method, a center frequency method of double thresholds is proposed to determine the value of the decomposition scale,  $k$ , in this paper. The center frequency method of double thresholds is a symmetry method. Based on the center frequency, the double thresholds of  $T_1$  and  $T_2$  are set. The threshold,  $T_1$ , is used to measure the difference between the maximum value of the central frequency of the corresponding modes under different scales and the maximum value of center frequencies of all modes. It is known by the center frequency method that the center frequency of the obtained IMF component under a certain scale is distributed from low frequency to high frequency. If the difference is greater than  $T_1$ , it indicates that the maximum IMF component of the central frequency in the original signal is not decomposed. The threshold,  $T_2$ , is used to measure the difference between the value of the central frequency of the  $L$ th IMF component and the value of the central frequency of the  $(L - 1)$ th IMF component under the same decomposition scale,  $k$  ( $k = L$ ). The VMD can effectively decompose the two pure harmonic signals with similar frequencies, and the IMF components with similar frequencies are retained according to the threshold,  $T_2$ . Set the maximum value,  $k_{\max}$ , of the decomposition scale,  $k$ . The signal is decomposed under decomposition scale,  $k = 1, 2, 3, \dots, k_{\max}$ , to obtain the value and the maximum value of the center frequency of the corresponding modes, and the maximum value of the center frequency of all modes under different scales. If the difference between the maximum value of the central frequency of the IMF component under a certain scale,  $k$ , and the maximum value of the central frequency of the IMF component under a certain scale,  $k + 1$ , is less than  $T_1$ , the difference between the value of the center frequency of the  $L$ th IMF component and the value of the center frequency of the  $(L - 1)$ th IMF component under the decomposition scale,  $k$ , is calculated ( $k = L$ ). If the difference value is less than the threshold,  $T_2$ , the minimum value of  $k$  meeting the above conditions is regarded as the optimal mode number of the VMD decomposition. The expressions of  $T_1$  and  $T_2$  are described as follows:

$$T_1 = a * cf\_max_{IMF_k} \quad (15)$$

$$T_2 = b * cf\_max_{IMF_k} \quad (16)$$

where there is  $k = 1, 2, 3, \dots, k_{max}$ .

In a practical application, according to the different application backgrounds, the values of  $a$  and  $b$  are determined according to the experiences. Therefore, according to the practical application in this paper, there are  $a = 0.08$  and  $b = 0.15$ .

### 3.2.2. The Flow and Steps of the Center Frequency Method of Double Thresholds

The flow of the center frequency method of double thresholds is shown in Figure 9.

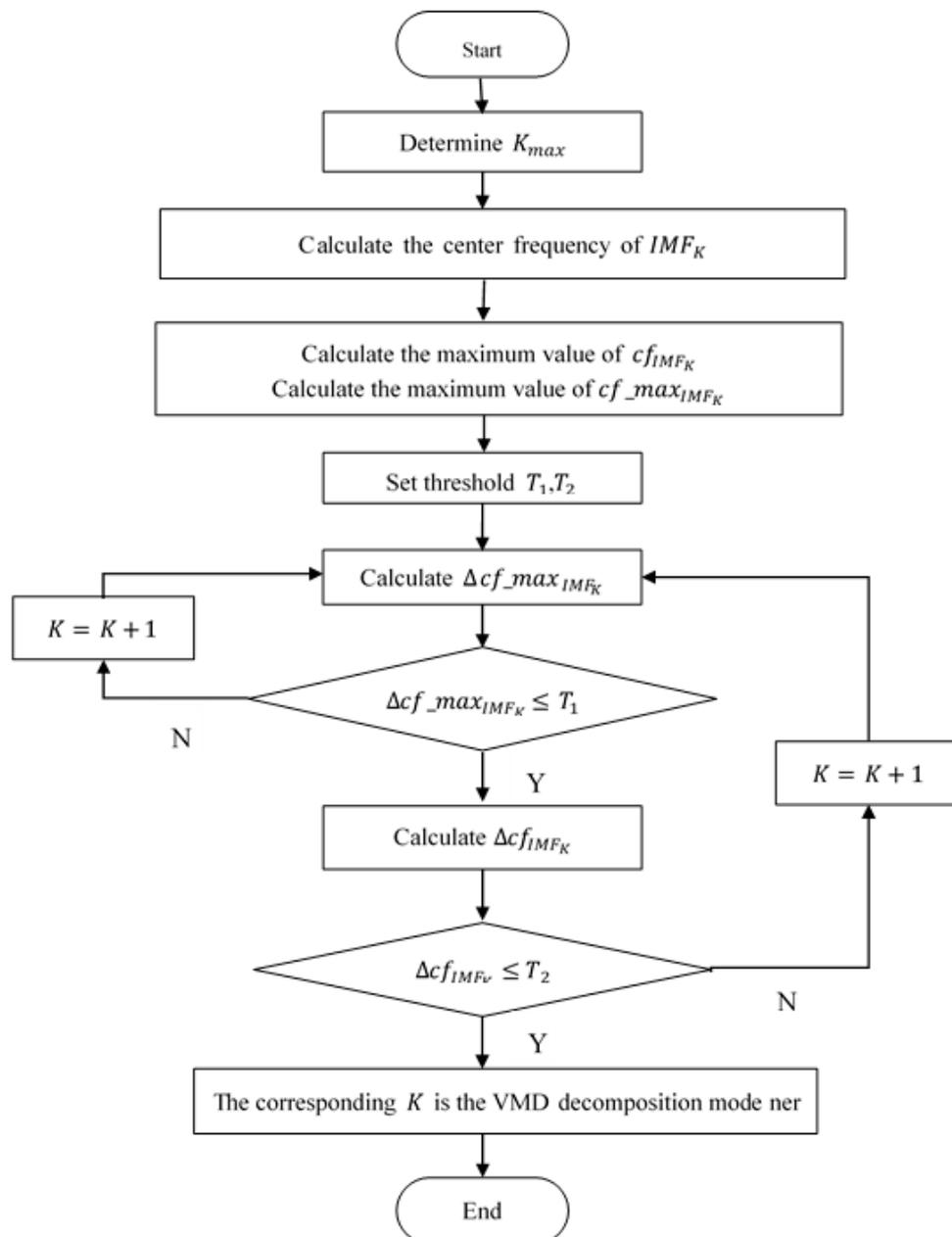


Figure 9. The flow of the center frequency method of double thresholds.

The implementation steps of the center frequency method of double thresholds are as follows:

**Step 1.** Determine the maximum value,  $k_{\max}$ , of the decomposition scale.

**Step 2.** The VMD method is used to decompose the original signal into a series of IMF components under different scales ( $IMF_k(k = 1, 2, 3, \dots, k_{\max})$ ). The central frequency values of each IMF component is calculated to obtain the set,  $cf_{IMF_k}(k = 1, 2, 3, \dots, k_{\max})$ .

**Step 3.** Calculate the maximum value of the center frequency of the set,  $cf_{IMF_k}$ , under different scales, as  $cf_{\max_{IMF_k}}(k = 1, 2, 3, \dots, k_{\max})$ .

**Step 4.** Calculate the maximum value of the center frequency of the set,  $cf_{\max_{IMF_k}}$ , as  $cf_{\max_{IMF}}$ .

**Step 5.** Set thresholds,  $a = 8\%$  and  $b = 15\%$ , for the expressions (15) and (16).

**Step 6.** Calculate the difference between the maximum center frequency value of all modes, as  $\Delta cf_{\max_{IMF_k}} = cf_{\max_{IMF}} - cf_{\max_{IMF_k}}(k = 1, 2, 3, \dots, k_{\max})$ .

**Step 7.** If there is  $\Delta cf_{\max_k} \leq T_1$ , Step 8 is executed. Otherwise, execute  $k = k + 1$  and go to Step 6.

**Step 8.** Calculate the difference value of the central frequency of the  $L$ th IMF component and the value of the central frequency of the  $(L - 1)$ th IMF component of the set,  $cf_{IMF}(k = L)$ , and is calculated as  $\Delta cf_{IMF} = cf_{IMF_{k_L}} - cf_{IMF_{k_{L-1}}}$ . If there is  $\Delta cf_{IMF} \leq T_2$ , the corresponding  $k$  ( $k = 1, 2, 3, \dots, k_{\max}$ ) of  $\Delta cf_{IMF}$  is the optimal mode number of the VMD method. Otherwise, execute  $k = k + 1$  and go to Step 6.

### 3.3. Effectiveness Analysis of the Center Frequency Method of Double Thresholds

The center frequency method of double thresholds is used to determine the number of VMD decomposition modes. When there is  $k = 2$ , the  $\Delta cf_{\max_{IMF_k}} \leq T_1$  is not be met. When there is  $k = 3, 4$ , the  $\Delta cf_{\max_{IMF_k}} \leq T_2$  is not be met. When there is  $k = 5$ , the condition of judgment is met at the same time. Therefore,  $k = 5$  is the optimal mode number of VMD decomposition. Compared with the center frequency method, the difference of the maximum value of the central frequency of the IMF component in the determined modes by two methods is very small, which shows that the component of the maximum value of the central frequency in the original signal has been decomposed. When there is  $k = 5$ , the central frequency value of two additional IMF components did not appear in  $k = 3$ . Therefore, the  $k = 5$  is better than  $k = 3$  according to the effective mode.

The time domain waveform and frequency domain waveform of the outer race fault vibration signal are shown in Figures 10 and 11.

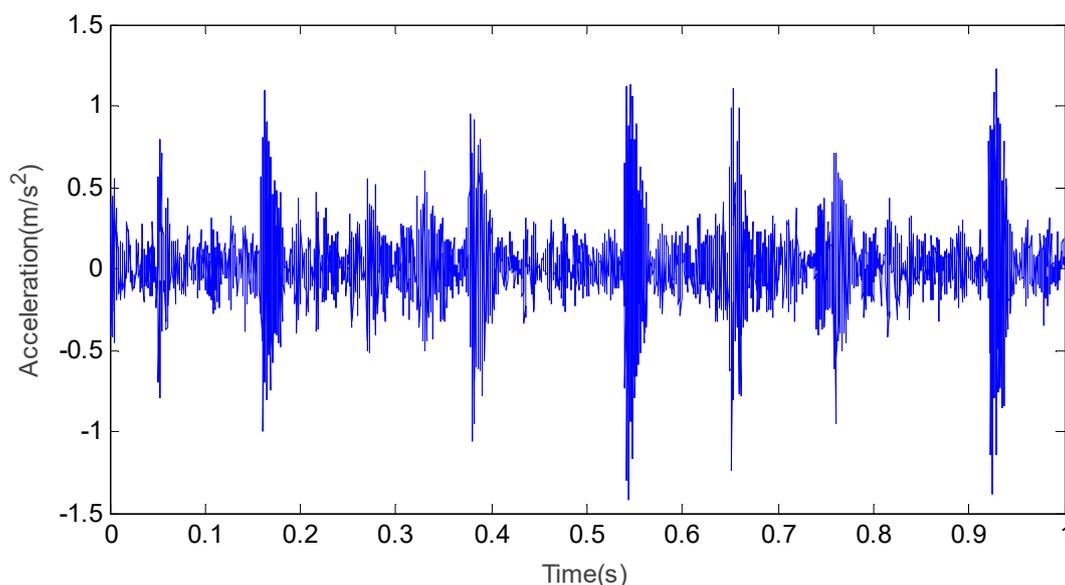
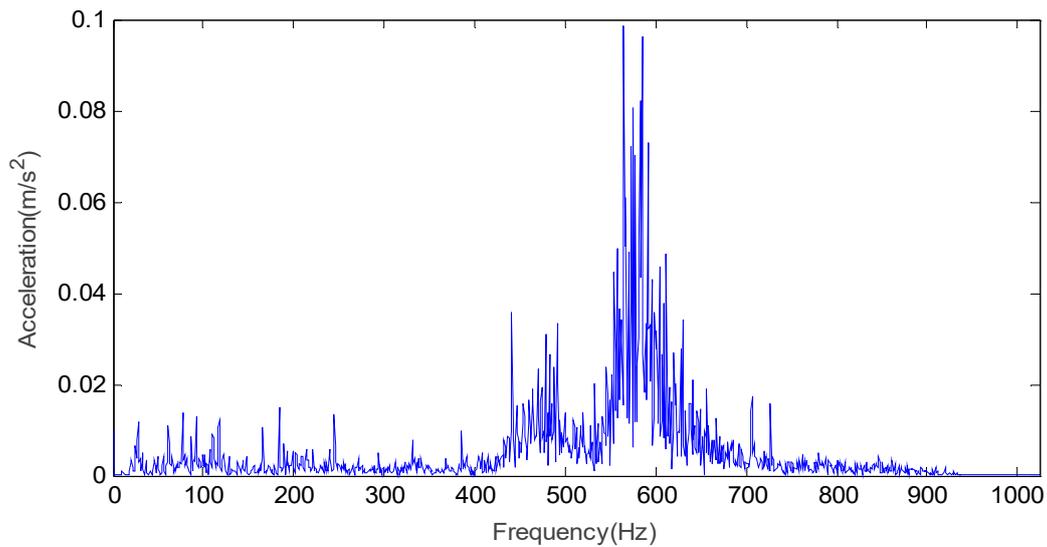
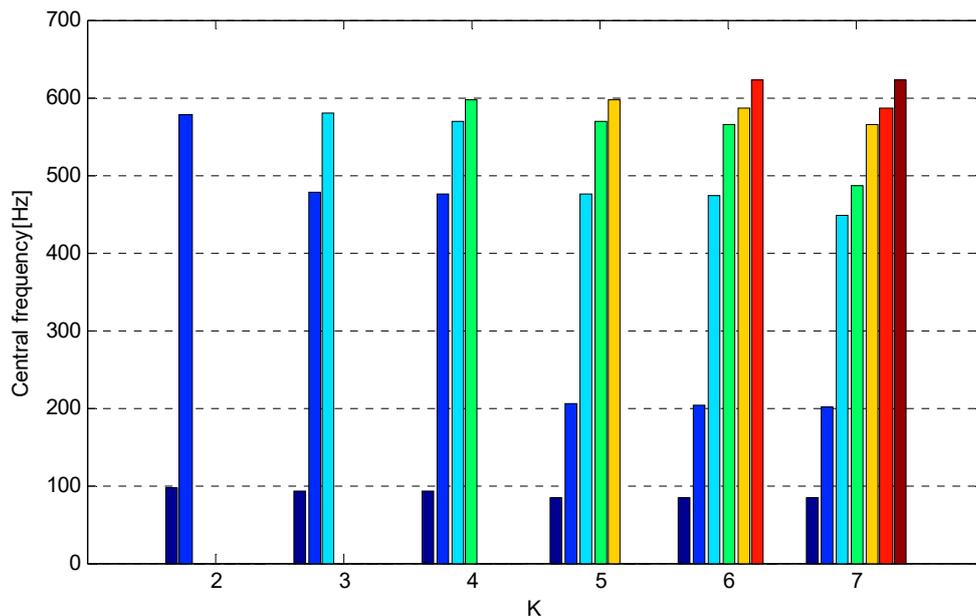


Figure 10. The time domain waveform of the outer race fault vibration signal.



**Figure 11.** The frequency domain waveform of the outer race fault vibration signal.

The outer race fault vibration signal is decomposed by VMD, and the central frequency value of each IMF component under different scales are shown in Figure 12.

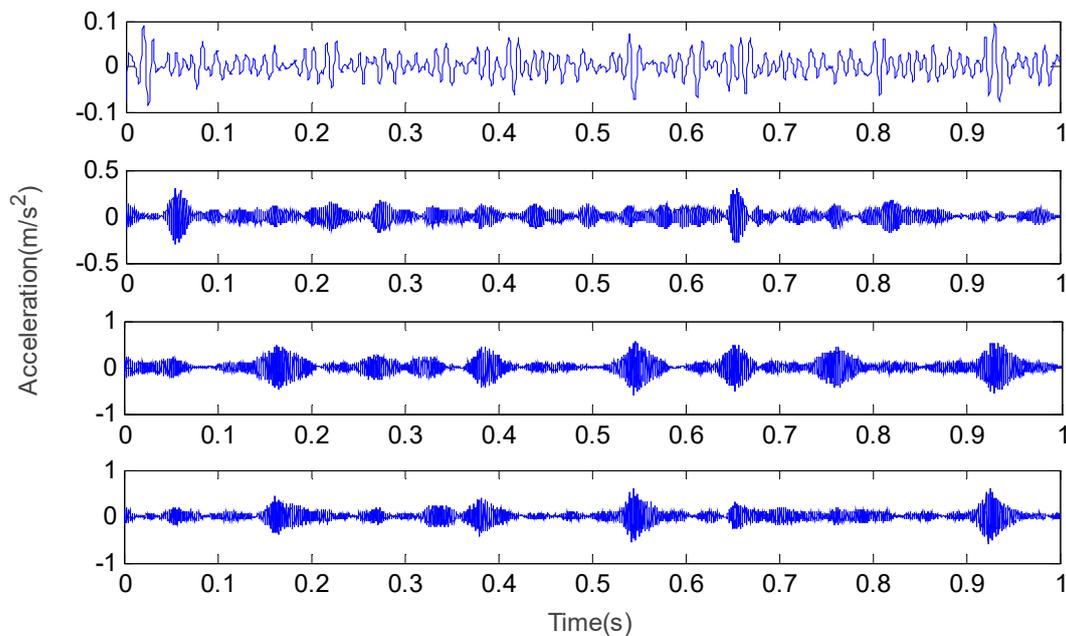


**Figure 12.** The central frequency values of each IMF component under different scales.

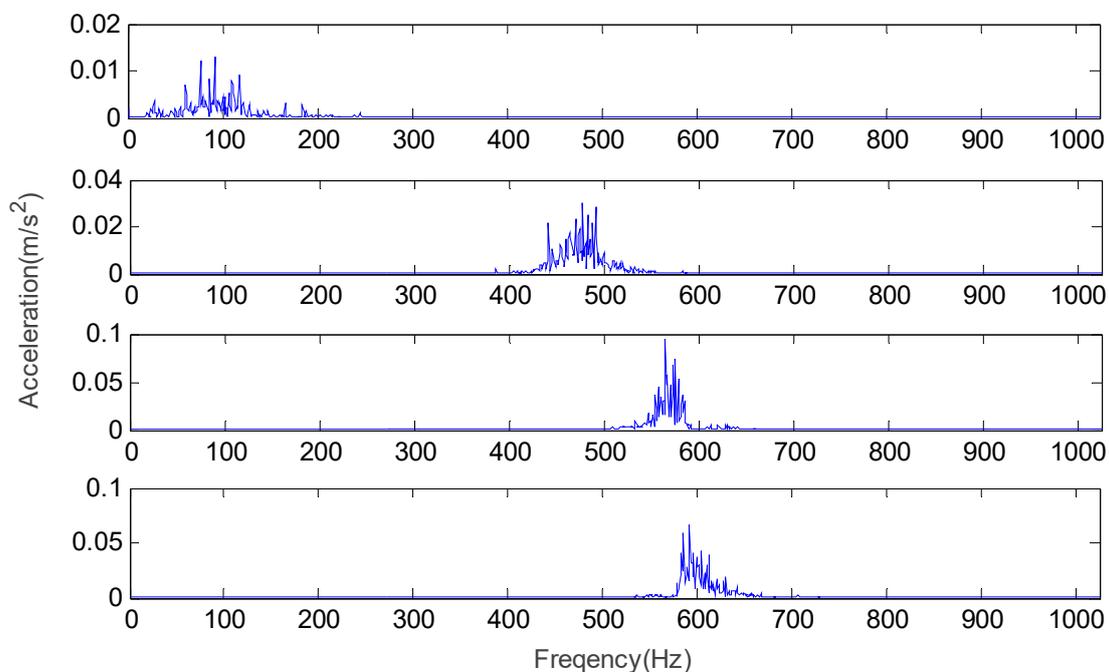
As can be seen from Figure 12, when there is  $k = 7$ , the center frequency value of the seventh IMF component obtains the maximum value of all IMF components. When there is  $k \geq 2$ , the difference between the maximum value of the central frequency of the IMF component and the maximum value of all IMF components is very small under different scales. According to the center frequency method, the value of  $k$  is determined as  $k = 2$ . When there is  $k \geq 3$ , the new frequency value appears in the mode. The analysis considers that the corresponding mode of the new frequency value is the effective mode. Therefore, when there is  $k = 2$ , the effective mode cannot be completely decomposed. The center frequency method of double thresholds is used to determine the number of VMD decomposition modes. When there is  $k = 2, 3$ , the  $\Delta cf_{\max_{IMF_k}} \leq T_2$  is not be met. When there is  $k = 4$ , the condition of judgment is met at the same time. Therefore,  $k = 4$  is the optimal mode number of VMD decomposition. Compared with the center frequency method, the difference of the maximum value

of the central frequency of the IMF component in the determined modes by the two methods is very small, which shows that the component of the maximum value of the central frequency in the original signal has been decomposed. When there is  $k = 4$ , the central frequency value of two additional IMF components did not appear in  $k = 2$ . Therefore, the  $k = 4$  is better than  $k = 2$  according to the effective mode.

When the value of  $k$  is 4, the time domain waveform and frequency domain waveform of the outer race fault vibration signal are shown in Figures 13 and 14.



**Figure 13.** The time domain waveform of each IMF for the outer race at  $k = 4$ .



**Figure 14.** The frequency domain waveform of each IMF for the inner race at  $k = 4$ .

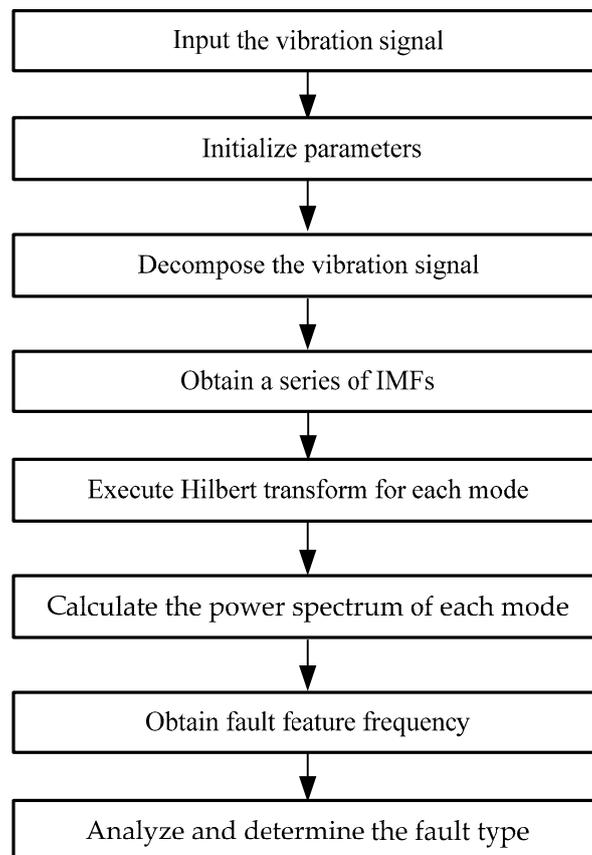
From the above experimental results and analysis, it can be found that the mode number of VMD with the central frequency method of double thresholds is superior to the mode number of VMD

with the central frequency method. In the VMD method, the central frequency method of double thresholds can effectively solve the problem that the person determines the value of  $k$  according to  $\Delta cf_{\max_k} \ll \Delta cf_{\max_{IMF_k}}$ . At the same time, the central frequency method of double thresholds makes up for the defect that the central frequency method cannot keep the IMF components with similar frequency values. Therefore, the central frequency method of double thresholds can determine the value of  $k$  in the maximum extent, which is the optimal mode number.

#### 4. Feature Extraction Method Based on the DTCFVMD and Hilbert Transform

##### 4.1. Feature Extraction Method

When the rolling bearings have a fault, its vibration signal contains a lot of running state information, which are non-stationary and multicomponent modulation signals. Due to the weak modulation source, early fault signals and noise interference from surrounding equipment and the environment, it is difficult to extract and identify the fault feature frequency. The VMD method is an adaptive and non-recursive signal decomposition method. It adopts a completely non-recursive method to realize the frequency domain decomposition of a signal, which has a strong decomposition ability, good anti-noise performance, and fast processing speed. Therefore, the proposed DTCFVMD method is used to decompose the nonlinear and non-stationary vibration signals of bearings to obtain a series of intrinsic mode functions under different scales. Then, the Hilbert transform is used to analyze the envelope of each mode component and calculate the power spectrum of each mode component, which is used to extract the fault feature frequency for determining the fault type of the motor bearing. The feature extraction model based on the DTCFVMD and Hilbert transform is shown in Figure 15.



**Figure 15.** The feature extraction model based on the DTCFVMD and Hilbert transform.

#### 4.2. Steps of Feature Extraction

The steps of the feature extraction model based on the DTCFVMD and Hilbert transform are described as follows:

**Step 1.** Preprocess the collected vibration signals of the motor rolling bearing, including de-noising, filtering, and so on.

**Step 2.** Propose a center frequency method of double thresholds to improve the VMD method for obtaining the adaptive VMD(DTCFVMD) method.

**Step 3.** The preprocessed vibration signal is decomposed by using the proposed DTCFVMD method to obtain a series of IMFs.

**Step 4.** Analyze the envelope of each mode component by using the Hilbert transform.

**Step 5.** Calculate the power spectrum of each mode component to extract the fault feature frequency.

**Step 6.** The fault feature frequency is compared with theoretical inherent fault frequency to accurately determine the fault type of the motor rolling bearing.

### 5. Verification and Results Analysis

#### 5.1. The Effectiveness Verification

The power spectrum represents the relationship between signal power and frequency; that is, the distribution of the signal power in the frequency domain. The Hilbert transform is used to solve the envelope of each IMF component, then the power spectrum of the envelope is solved. The maximum frequency value of the power spectral density is marked in order to express the fault characteristic frequency, rotating frequency, and frequency multiplication by power spectrum analysis.

According to the center frequency method of double thresholds, the optimal mode number of VMD decomposition for the inner race vibration signal and outer race vibration signal are 5 and 4, respectively. In order to verify the validity of the optimal mode, the signal power spectrum is used to extract the fault characteristic and estimate the effective frequency.

For the inner race fault vibration signal, the power spectrum of each IMF component under  $k = 4$ ,  $k = 5$ , and  $k = 6$  is shown in Figure 16 and Table 3.

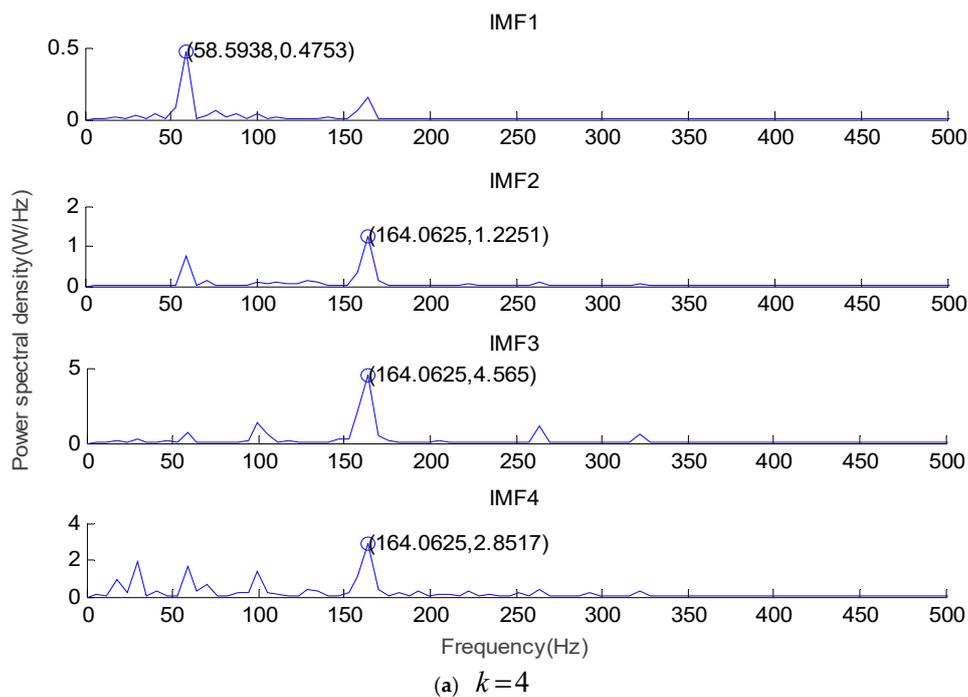


Figure 16. Cont.

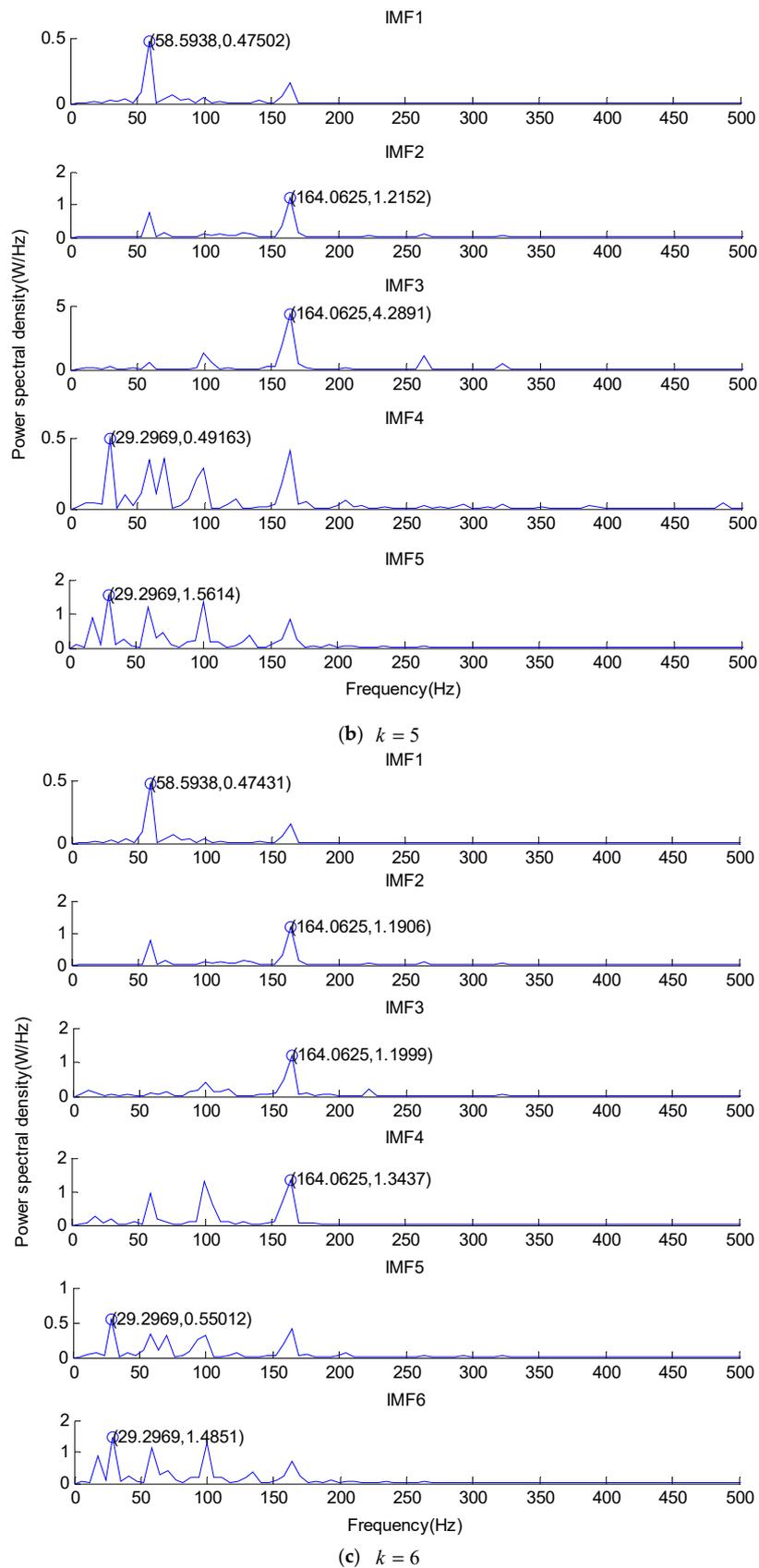


Figure 16. The power spectrum of each IMF component under (a)  $k = 4$ , (b)  $k = 5$ , and (c)  $k = 6$ .

**Table 3.** Frequency values of power spectrum of each IMF under  $k = 4, k = 5,$  and  $k = 6$ .

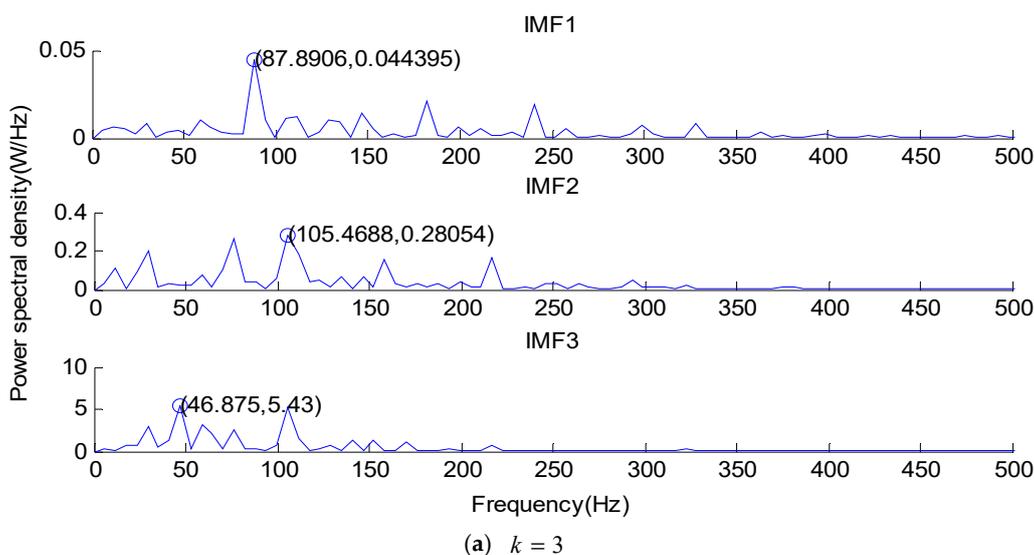
	IMF1 (Hz)	IMF2 (Hz)	IMF3 (Hz)	IMF4 (Hz)	IMF5 (Hz)	IMF6 (Hz)
$k = 4$	58.6	164.1	164.1	164.1		
$k = 5$	58.6	164.1	164.1	29.3	29.3	
$k = 6$	58.6	164.1	164.1	164.1	29.3	29.3

As can be seen from Figure 16 and Table 3, 29.3 Hz represents the rotating frequency, 58.6 Hz represents the frequency doubling of the rotating frequency, and 164.1 Hz represents the fault characteristic frequency of the inner race. When there is  $k = 5$ , the rotating frequency can be obtained from the frequency spectrum, but the rotating frequency cannot be obtained from the frequency spectrum when there is  $k = 4$ . When there are  $k = 5$  and  $k = 6$ , the rotating frequency, the frequency doubling of the rotating frequency, and the fault characteristic frequency of the inner race can be obtained from the frequency spectrum. When there is  $k = 6$ , it has one more fault characteristic frequency at 164.1 Hz than  $k = 5$ . However, the frequency value repeats, so  $k = 5$  is the optimal mode number of the VMD decomposition.

For the outer race vibration signal, the power spectrum of each IMF component under  $k = 4, k = 5,$  and  $k = 6$  is shown in Figure 17 and Table 4.

As can be seen from Figure 17 and Table 4, 29.3 Hz represents the rotating frequency, 87.9 Hz represents the frequency tripling of the rotating frequency, 105.5 Hz represents the fault characteristic frequency of the outer race, and 46.9 Hz represents the difference between the fault characteristic frequency and the frequency doubling of the rotating frequency of the outer race. When there is  $k = 4$ , the rotating frequency can be obtained from the frequency spectrum, but the rotating frequency cannot be obtained from the frequency spectrum when there is  $k = 3$ . When there are  $k = 4$  and  $k = 5$ , the frequency tripling of the rotating frequency, the difference between the fault characteristic frequency, and the frequency doubling of the rotating frequency of the outer race can be obtained from the frequency spectrum. When there is  $k = 5$ , it is not able to obtain the rotating frequency. Therefore,  $k = 4$  is the optimal mode number of the VMD decomposition.

The frequency value of each IMF component extracted from the power spectrum shows that the value of  $k$  determined by the central frequency method of double thresholds is the optimal mode number of the VMD method. Therefore, it is proved that the proposed central frequency method of double thresholds is effective.



**Figure 17.** Cont.

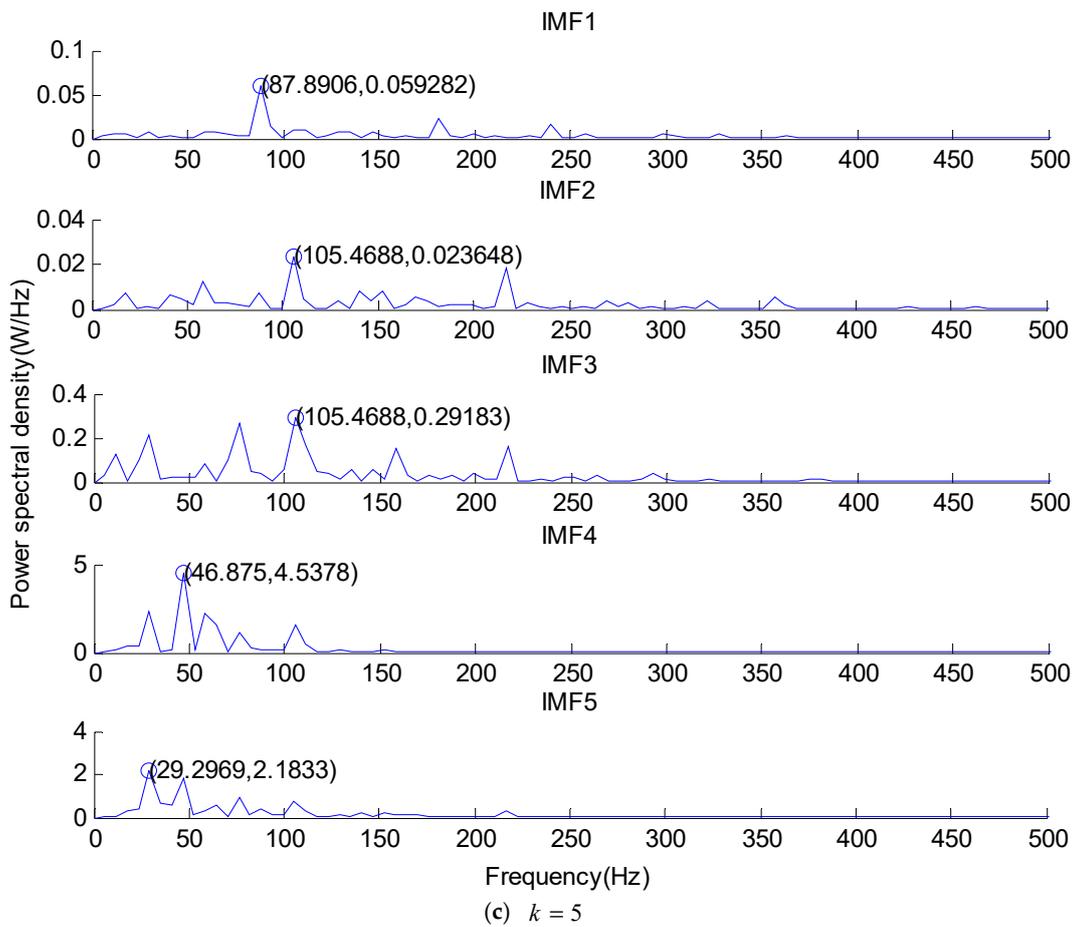
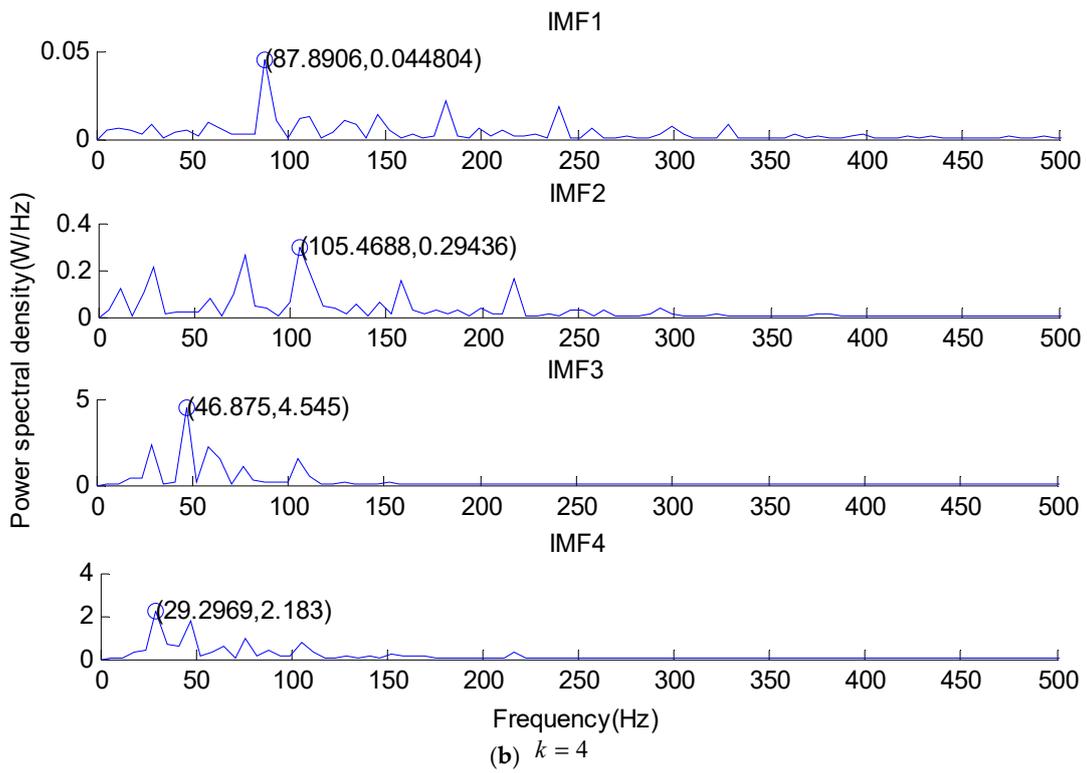


Figure 17. The power spectrum of each IMF component under (a)  $k = 3$ , (b)  $k = 4$ , and (c)  $k = 5$ .

**Table 4.** Frequency values of power spectrum of each IMF component under  $k = 3$ ,  $k = 4$ , and  $k = 5$ .

	IMF1 (Hz)	IMF2 (Hz)	IMF3 (Hz)	IMF4 (Hz)	IMF5 (Hz)
$k = 3$	87.9	105.5	46.9		
$k = 4$	87.9	105.5	46.9	29.3	
$k = 5$	87.9	105.5	105.5	46.9	46.9

## 5.2. Comparison and Analysis

To verify the effectiveness of VMD with the center frequency method of double thresholds, the VMD with the center frequency method of double thresholds (DTCFVMD) is compared with the EMD and EEMD methods. The DTCFVMD, EMD, and EEMD methods are used to decompose the vibration signals of the inner ring and outer ring to obtain a series of IMFs. Then, the Hilbert transform is used to obtain the power spectrum of the decomposed IMF components. The effectiveness of DTCFVMD is explained by analyzing the effective frequency components extracted from the power spectrum.

The frequency values extracted by the power spectrum of each IMF component of the inner race vibration signal by using the DTCFVMD ( $k = 5$ ), EMD, and EEMD methods are shown in Table 5.

**Table 5.** Frequency values extracted by the power spectrum of each IMF component of the inner race vibration signal.

	M1 (Hz)	M2 (Hz)	M3 (Hz)	M4 (Hz)	M5 (Hz)	M6 (Hz)	M7 (Hz)	M8 (Hz)	M9 (Hz)	M10 (Hz)	M11 (Hz)	M12 (Hz)
EMD	164.1	164.1	164.1	29.3	23.4	41.0	5.9	5.9	5.9	5.9	5.9	
EEMD	164.1	164.1	164.1	58.6	35.2	29.3	5.9	11.7	11.7	5.9	5.9	5.9
DTCFVMD	58.6	164.1	164.1	29.3	29.3							

In Table 5, 29.3 Hz represents the rotating frequency, 58.6 Hz represents the frequency doubling of the rotating frequency, and 164.1 Hz represents the fault characteristic frequency of the inner race. As can be seen from Table 5, the number of effective frequencies of the power spectrum in the IMF component by using the EEMD method is more than that of the EMD method. Therefore, the decomposition effect of the EEMD method is better than that of the EMD method. Compared with the DTCFVMD method, the number of effective frequencies of IMF components by using the EEMD method is less than that of the EMD method and the EEMD method, and each mode can extract the effective frequency value. However, there exists some invalid frequency values in the modes of the EMD decomposition and the EEMD decomposition. In general, the decomposition effect of the DTCFVMD method is better than the decomposition effect of the EMD method and the EEMD method.

The frequency values extracted by the power spectrum of each IMF component of the outer race vibration signal by using the DTCFVMD ( $k = 4$ ), EMD, and EEMD methods are shown in Table 6.

**Table 6.** Frequency values extracted by the power spectrum of each IMF component of the outer race vibration signal.

	M1 (Hz)	M2 (Hz)	M3 (Hz)	M4 (Hz)	M5 (Hz)	M6 (Hz)	M7 (Hz)	M8 (Hz)	M9 (Hz)	M10 (Hz)	M11 (Hz)	M12 (Hz)
EMD	105.5	105.5	111.3	87.9	17.6	23.4	11.7	11.7	11.7	5.9		
EEMD	105.5	105.5	105.5	105.5	87.9	29.3	52.7	5.9	5.9	5.9	5.9	5.9
DTCFVMD	87.9	105.5	46.9	29.3								

In Table 6, 29.3 Hz represents the rotating frequency, 87.9 Hz represents the frequency tripling of the rotating frequency, 105.5 Hz represents the fault characteristic frequency of the outer race, 52.7 Hz represents the fault characteristic frequency of 1/2, 11.7 Hz represents the fault characteristic frequency of 1/9, and 46.9 Hz represents the difference between the fault characteristic frequency and the frequency doubling of the rotating frequency of the outer race. As can be seen from Table 6,

the number of effective frequencies of the power spectrum in the IMF component by using the EEMD method is more than that of the EMD method, and the EEMD method can obtain the rotating frequency. Therefore, the decomposition effect of the EEMD method is better than that of the EMD method. Compared with the DTCFVMD method, the number of effective frequencies of IMF components by using the EEMD method is less than that of the EMD method and the EEMD method, and each mode can extract the effective frequency value. However, there exists some invalid frequency values in the modes of the EMD decomposition and the EEMD decomposition. In general, the decomposition effect of the DTCFVMD method is better than the decomposition effect of the EMD method and the EEMD method.

## 6. Conclusions

Time-frequency analysis methods are the most commonly applied methods. They are very significant to deeply analyze vibration signals for monitoring the health state of rotating machinery. The VMD is an adaptive and non-recursive signal decomposition method, which can decompose a real valued input signal into an ensemble of sub-signals or modes with specific sparsity properties. However, it has the disadvantage of the experienced selection mode number. Therefore, a center frequency method of double thresholds was proposed to determine the decomposed mode number of the VMD method. It can adaptively obtain the optimal mode number that matches the analyzed signal, and decompose the vibration signal into a series of IMFs under different scales. The Hilbert transform method was used to calculate the power spectrum of each mode component to determine the fault type. The actual fault vibration signal of the motor bearing was selected to test and verify the effectiveness of the proposed DTCFVMD method. The experiment results demonstrate that the proposed DTCFVMD method can determine the optimal value of  $k$  by using the central frequency method of double thresholds for the inner race and outer race of the motor bearing. The proposed DTCFVMD method was compared with the VMD method, EMD, and EEMD methods for verifying the effectiveness of vibration analysis and feature extraction. Therefore, the proposed DTCFVMD method is an effective method to analyze the vibration signal. It can provide a new idea to improve the VMD method for analyzing the vibration analysis and extracting fault features of rotating machinery.

Due to the ineffectiveness of the proposed DTCFVMD method for analyzing the vibration signal of the rolling element of motor bearings, and the equipment and working conditions becoming more complex, the VMD method is needed to be further deeply studied. In future work, the new methods are proposed to analyze the vibration signal of different equipment, conditions, and industries.

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**Conflicts of Interest:** The authors declare no conflict of interest.

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